

Design of High Energy Absorbing Materials Based on Porous Superelastic NiTi

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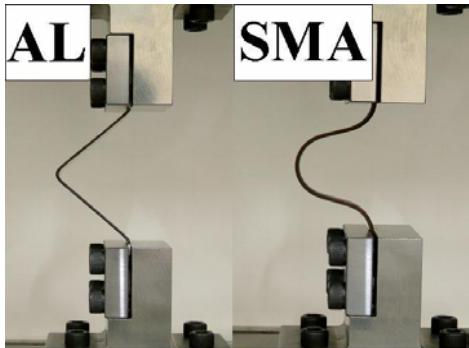
Applications of porous shape memory alloys (SMAs)

- High energy absorbing Structure and damping devices
- Medical implant
- Cooling surface

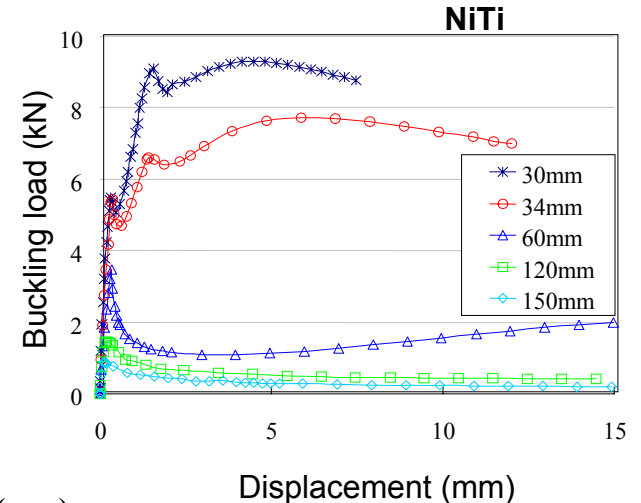
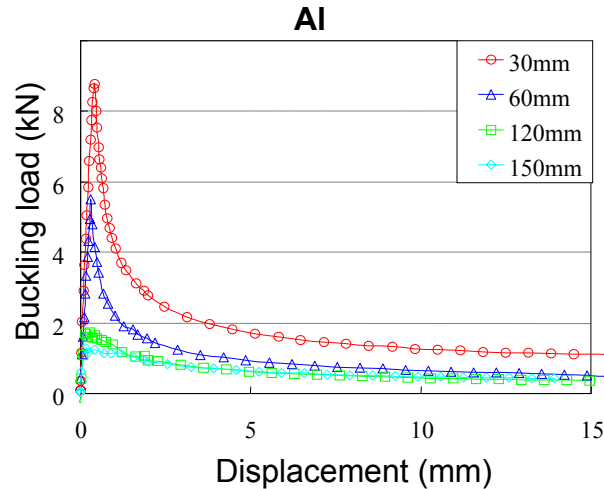


Background

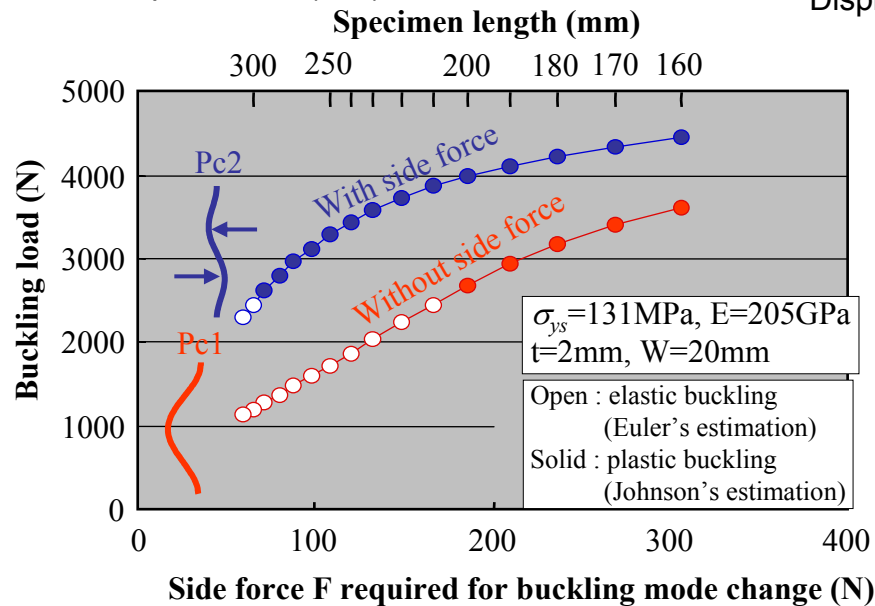
(A) Shape memory alloys such as NiTi exhibit larger buckling load as compared with other structural materials (for example aluminum) (Suzuki, Urushiyama and Taya, 2004)



Test specimens after buckling

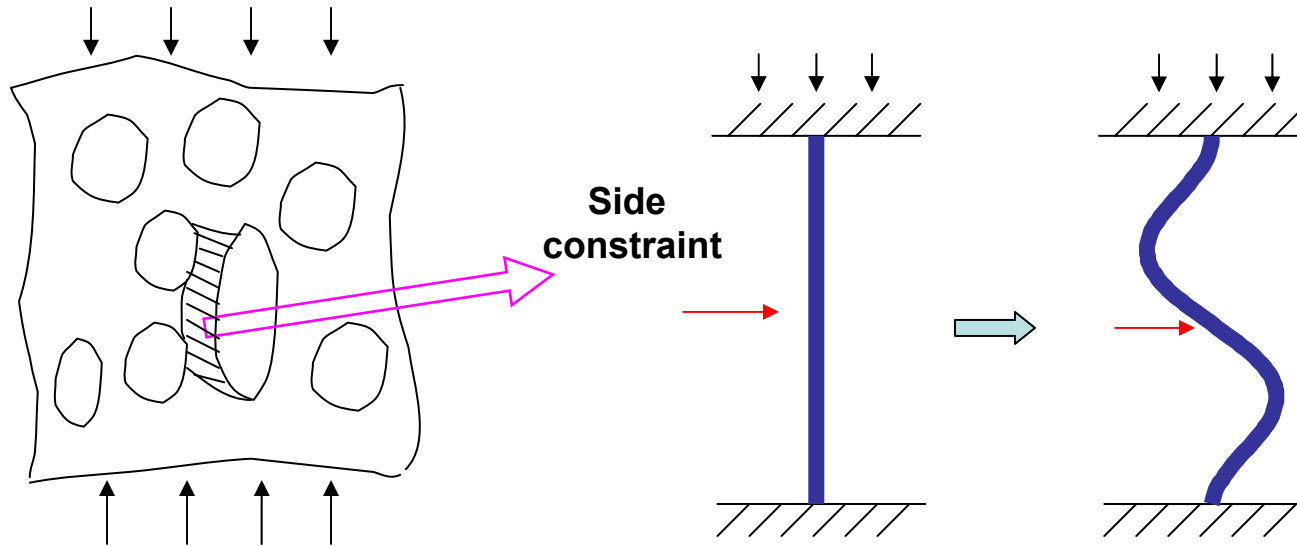


(B) Increase in buckling load by applying side constraint



Approach to Design Porous SMA

- (1) Combine previous results of (A) and (B) in Background, to design a porous SMA such as NiTi which exhibits large energy absorption capacity per weight.



Micro-pillar with side constraint

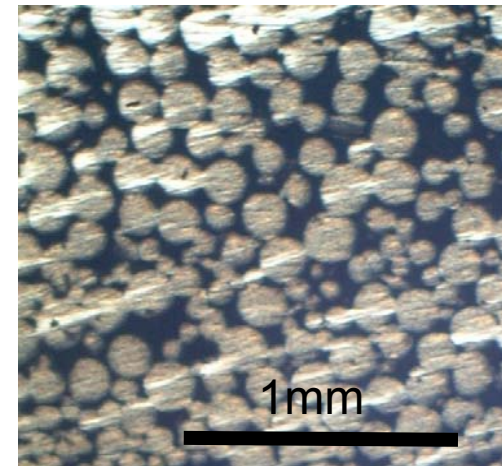
- (2) Use the large stress-strain characteristics of SMA.
- (3) Design composite structure made of porous NiTi and NiTi spring

Porous NiTi SMA fabricated by Spark Plasma Sintering (SPS)

Specimen	Porosity (%)	Spark Plasma Processing Condition	Transformation temperature (° C)
Solid NiTi	0	950°C under 50MPa, 5 minutes	$A_s = 18.888$ $A_f = 37.182$
13% porous NiTi	13	800°C under 25MPa, 5 minutes	$A_s = 1.288$ $A_f = 23.823$

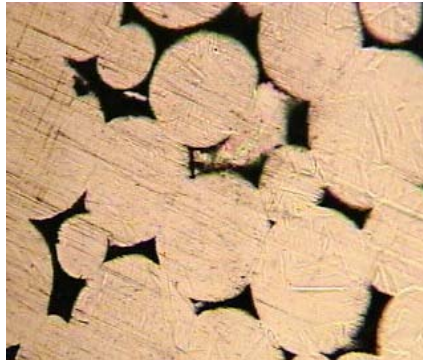


SPS equipment

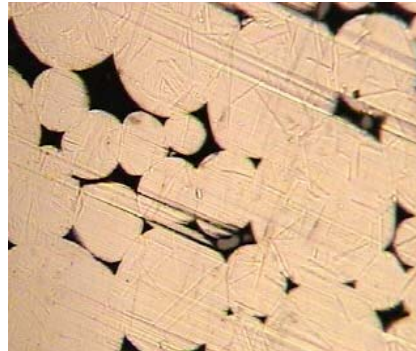


Uniform microstructure of
13% porosity NiTi

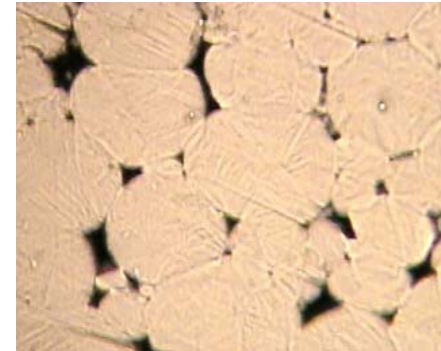
Compression tests under room temperature



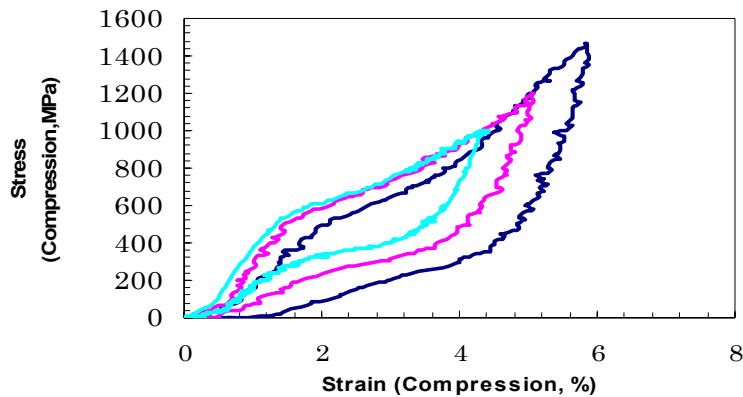
(a) Before Compression



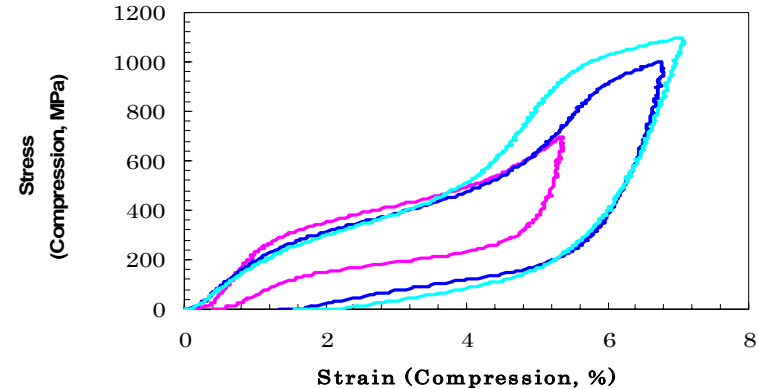
(b) Compression up to 5% and unloaded



(c) Compression up to 7% and unloaded

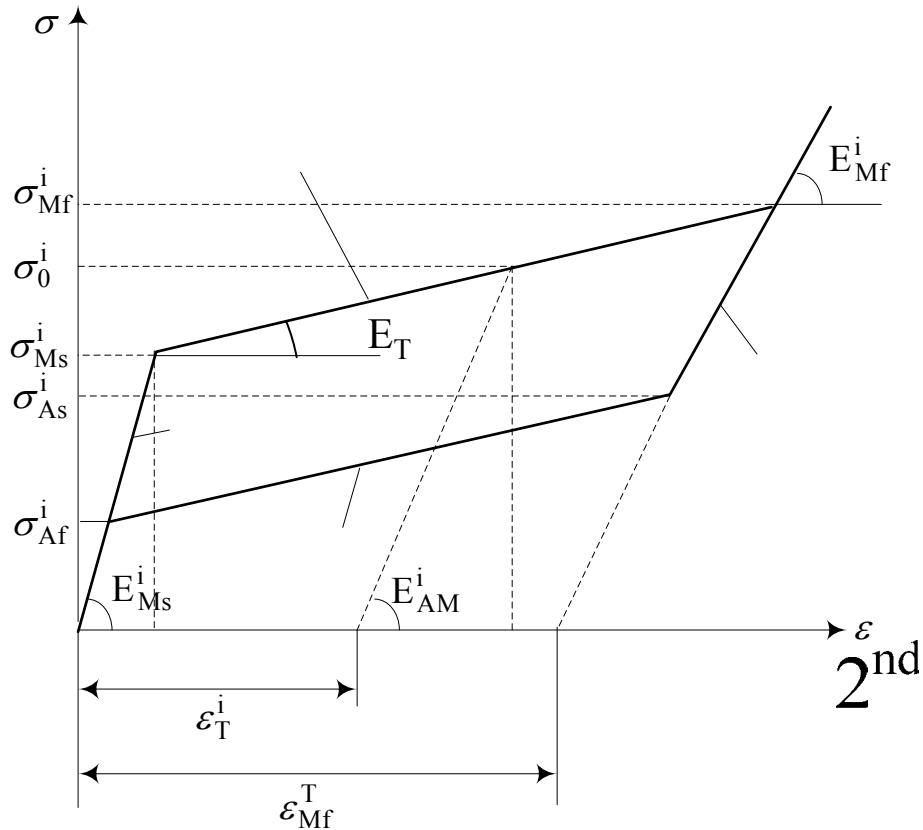


Superelastic response of solid NiTi under static compression



Superelastic response of porous NiTi under static compression ($f_p=13\%$)

Idealized Stress-Strain Curve



1st Stage

100% austenite, E_{Ms}

2nd Stage

Austenite 100% \rightarrow 0%

Martensite 0% \rightarrow 100%

3rd Stage

100% martensite, E_{Mf}

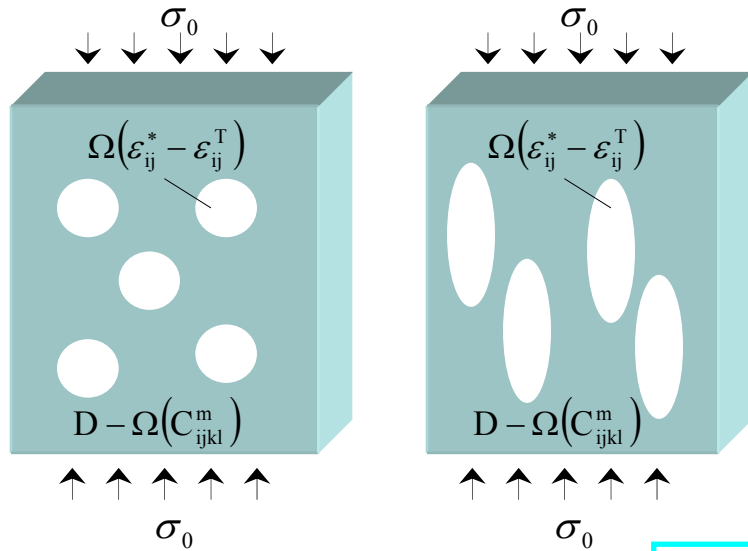
2nd stage

Austenite 0% \rightarrow 100%

Martensite 100% \rightarrow 0%

C_i

Model-1: Stress-strain curve of NiTi with closed pores

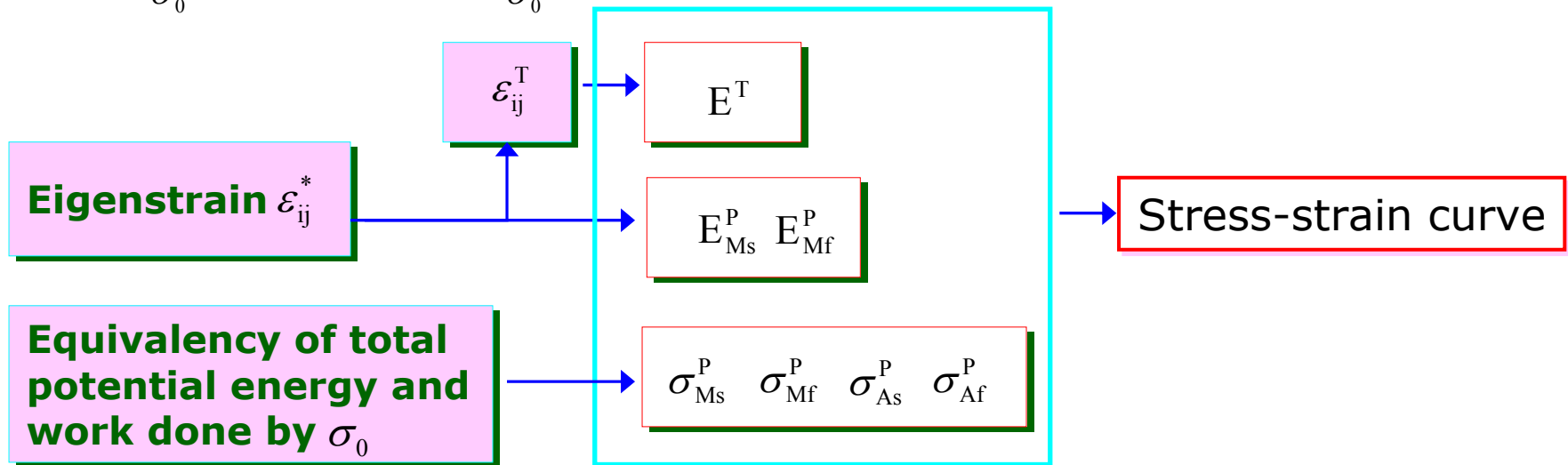


Eshelby's equivalent inclusion method

$$\begin{aligned}\sigma_{ij}^0 + \sigma_{ij} &= C_{ijkl}^m \left[e_{kl}^0 + \bar{e}_{kl} + e_{kl} - (\varepsilon_{kl}^* - \varepsilon_{kl}^T) \right] \\ &= C_{ijkl}^m \left(\varepsilon_{kl}^0 + \bar{\varepsilon}_{kl} + \varepsilon_{kl} - \varepsilon_{kl}^{**} \right) \\ &= C_{ijkl}^p \left(\varepsilon_{kl}^0 + \bar{\varepsilon}_{kl} + \varepsilon_{kl} \right)\end{aligned}$$

$$\sigma_{ij}^0 = C_{ijkl}^m \varepsilon_{kl}^0 \quad \varepsilon_{kl} = S_{klmn} \varepsilon_{mn}^{**}$$

$$\varepsilon_{kl}^{**} = -\frac{1}{1 - f_p} (S_{klmn} - I)^{-1} C_{ijkl}^{m-1} \sigma_{ij}^0$$



Model-1: Stress-strain curve of NiTi with closed pores

Critical stresses

$$\text{Bi: } \sigma_{Ms}^P = (1 - f_p) \sigma_{Ms}^S$$

$$\text{Di: } \sigma_{Mf}^P = (1 - f_p) \sigma_{Mf}^S$$

$$\text{di: } \sigma_{As}^P = (1 - f_p) \sigma_{As}^S$$

$$\text{bi: } \sigma_{Af}^P = (1 - f_p) \sigma_{Af}^S$$

Linear stage (equivalency of strain energy density)

$$\frac{1}{2} C_{ijkl}^{c-1} \sigma_{ij}^0 \sigma_{kl}^0 = \frac{1}{2} C_{ijkl}^{m-1} \sigma_{ij}^0 \sigma_{kl}^0 + \frac{1}{2} f_p \sigma_{ij}^0 \epsilon_{kl}^{**}$$

Stiffness

1st stage

$$\frac{E_{Ms}}{E_A} = \frac{1}{1 + \eta f_p}$$

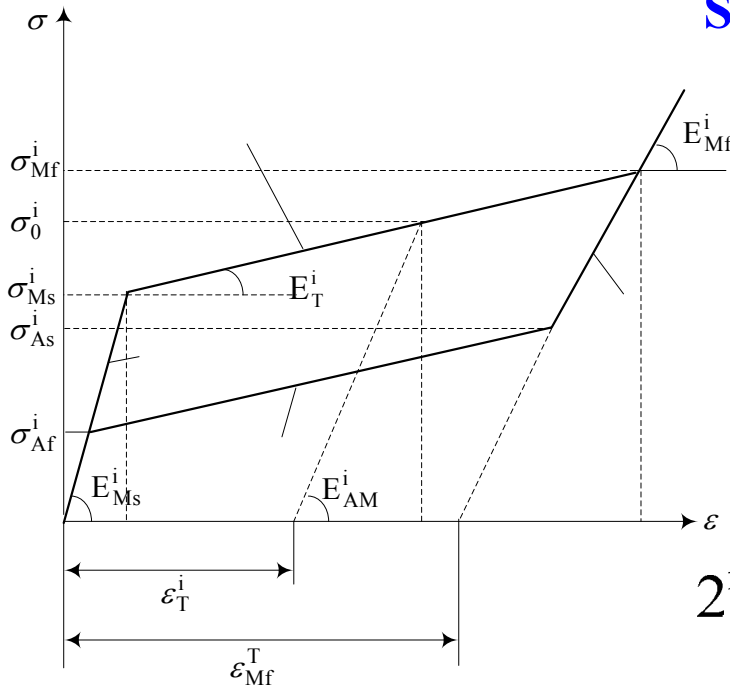
3rd stage

$$\frac{E_{Mf}}{E_M} = \frac{1}{1 + \eta f_p}$$

$$\left(\text{where } \eta = -\frac{1}{1 - f_p} (S_{3333} - 1)^{-1} \right)$$

Model-1: Stress-strain curve of NiTi with closed pores

Stress-induced martensitic transformation



Macroscopic strain energy density of porous NiTi

$$W_{\text{macro}} = \frac{1}{2} \left(\sigma_{\text{Ms}}^{\text{P}} + \sigma_0^{\text{P}} \right) \left(\varepsilon_{\text{T}} + \frac{\sigma_0^{\text{P}}}{E_{\text{AM}}} - \frac{\sigma_{\text{Ms}}^{\text{P}}}{E_{\text{Ms}}} \right)$$

Microscopic strain energy density (Taya et al, 1991)

$$W_{\text{micro}} = \frac{1}{2} \mathbf{C}_{ijkl}^{m-1} \sigma_{ij}^0 \sigma_{kl}^0 + \frac{1}{2} f_{\text{P}} \sigma_{ij}^0 \varepsilon_{kl}^* (\varepsilon_{\text{T}})$$

2nd Equating the above two equations given as
stage

$$A \varepsilon_{\text{T}}^2 + B \varepsilon_{\text{T}} + C = 0 \quad \rightarrow \quad \varepsilon_{\text{T}} = \frac{-B + \sqrt{B^2 - 4AC}}{2A}$$

Young's modulus

$$E_{\text{T}} = \frac{\sigma_0^{\text{P}} - \sigma_{\text{Ms}}^{\text{P}}}{\varepsilon_{\text{T}}}$$

(2nd and 4th Stage)

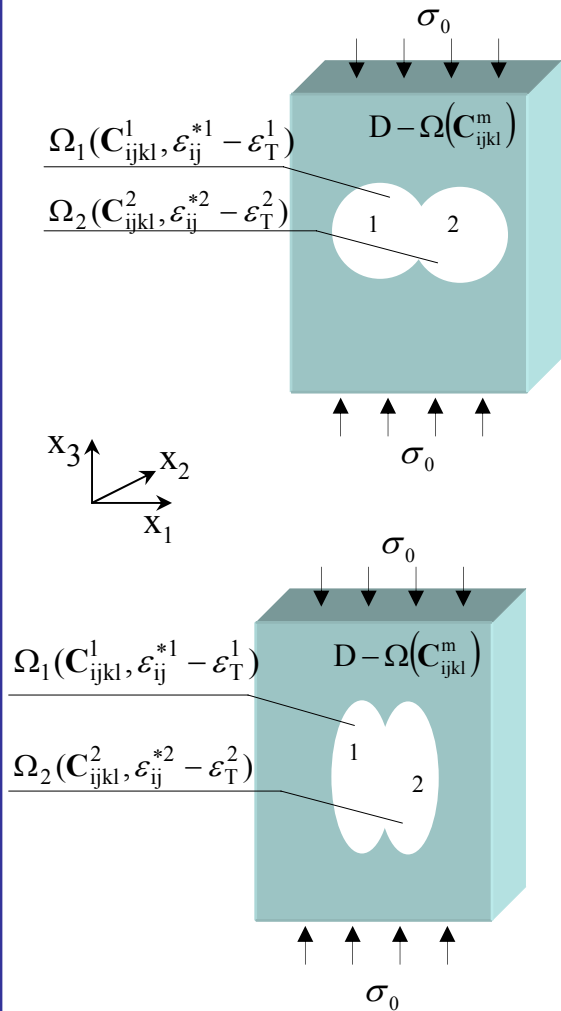
d_i 3rd stage

Assume the Young's modulus as the function of transformation strain

$$E_{\text{AM}}^i = E_{\text{Ms}}^i - \frac{E_{\text{Ms}}^i - E_{\text{Mf}}^i}{\varepsilon_{\text{Mf}}^i - \sigma_{\text{Mf}}^i / E_{\text{Mf}}^i} \varepsilon_{\text{T}}^i$$

1st stage

Model-2: Stress-strain curve of NiTi with open pores



Disturbance stress σ_{ij}^1

Disturbance stress σ_{ij}^2

Eigenstrain ϵ_{ij}^{*1} in Ω_1

Eigenstrain ϵ_{ij}^{*2} in Ω_2

Equivalency of total potential energy and work done by σ_0

Average eigenstrain ϵ_{ij}^*

ϵ_{ij}^T

σ_{Ms}^P σ_{Mf}^P σ_{As}^P σ_{Af}^P

E_{Ms}^P E_{Mf}^P

E^T

Model-2: Stress-strain curve of NiTi with open pores

Ratio of Eigenstrain in Ω_2 to Ω_1

$$\varepsilon_{33}^{*2} = \xi \varepsilon_{33}^{*1}$$

$$\xi = -\frac{(1+\nu)(1-2\nu)}{(1-\nu)^2(S_{3333}-1)}$$

Average eigenstrain

$$\varepsilon_{33}^{*(I/III)} = \frac{1}{2}(1+\xi)\varepsilon_{33}^{*1}$$

$$\varepsilon_{33}^{*(II/IV)} = \frac{1}{2}(\varepsilon_T^1 + \varepsilon_T^2) - \frac{1}{2}(1+\xi)\varepsilon_{33}^{*1}$$

Transformation strain

$$\varepsilon_T = \frac{-B_2 + \sqrt{(B_2)^2 - 4A_2C_2}}{2A_2}$$

Stiffness

1st stage

$$\frac{E_{Ms}}{E_A} = \frac{1}{1+\eta f_p}$$

2nd stage and
4th Stage

$$E_T = \frac{\sigma_0^P - \sigma_{Ms}^P}{\varepsilon_T}$$

3rd stage

$$\frac{E_{Mf}}{E_M} = \frac{1}{1+\eta f_p}$$

Critical stress

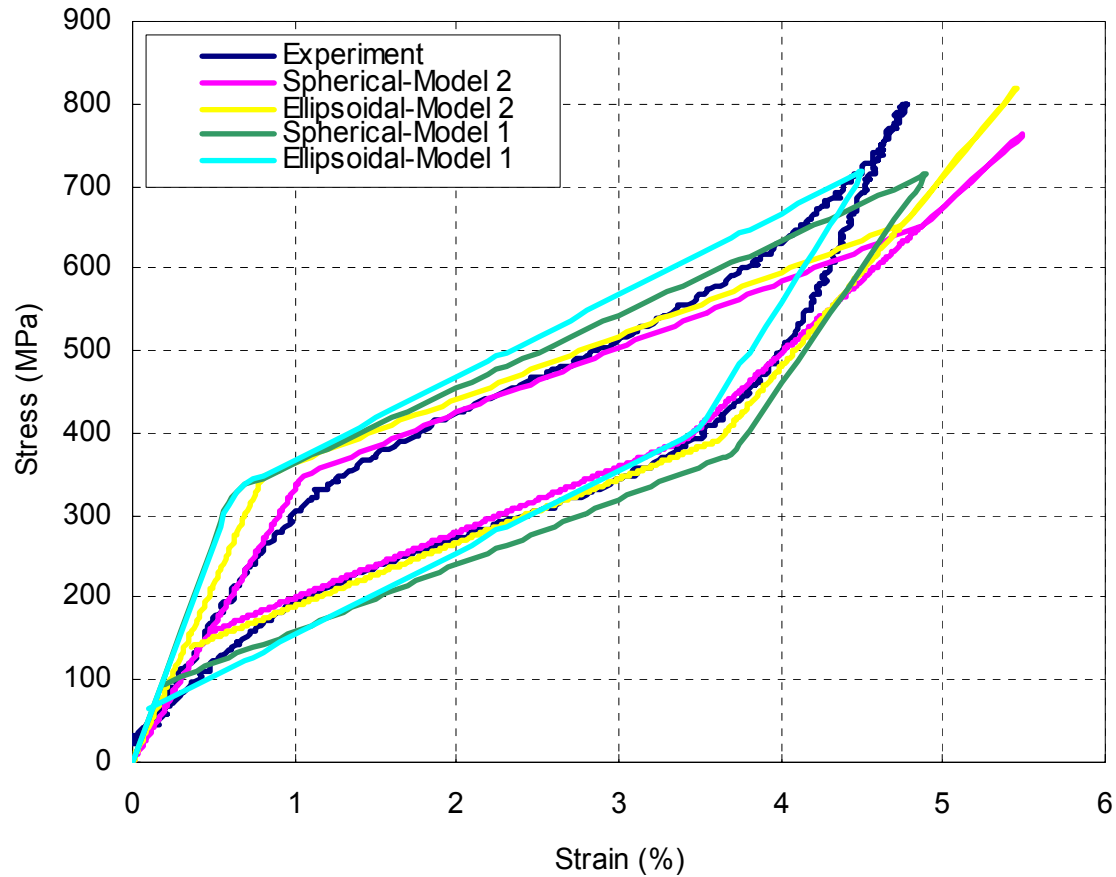
$$\sigma_{Ms}^P = (1-f_p)\sigma_{Ms}^S$$

$$\sigma_{Mf}^P = (1-f_p)\sigma_{Mf}^S$$

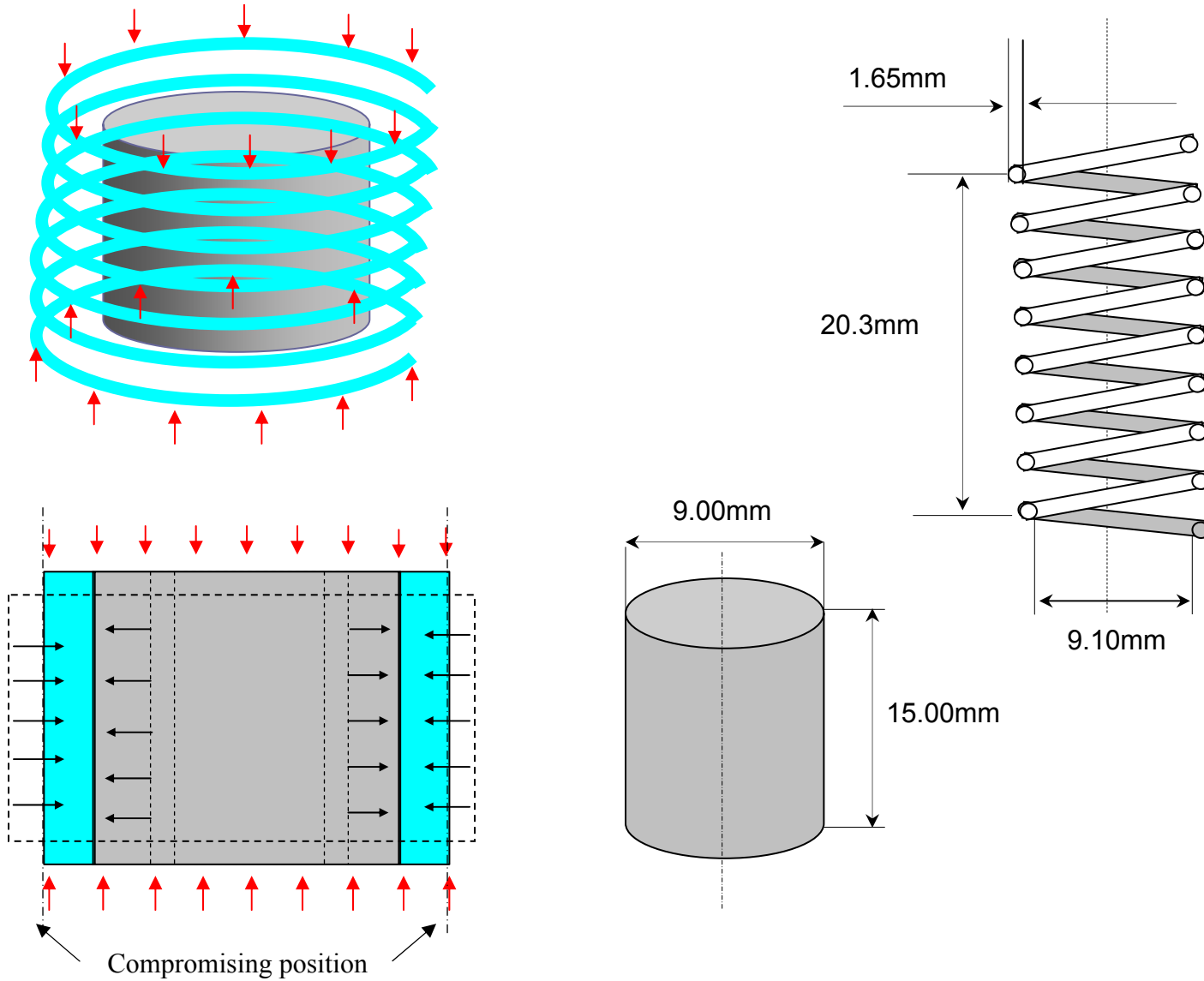
$$\sigma_{As}^P = (1-f_p)\sigma_{As}^S$$

$$\sigma_{Af}^P = (1-f_p)\sigma_{Af}^S$$

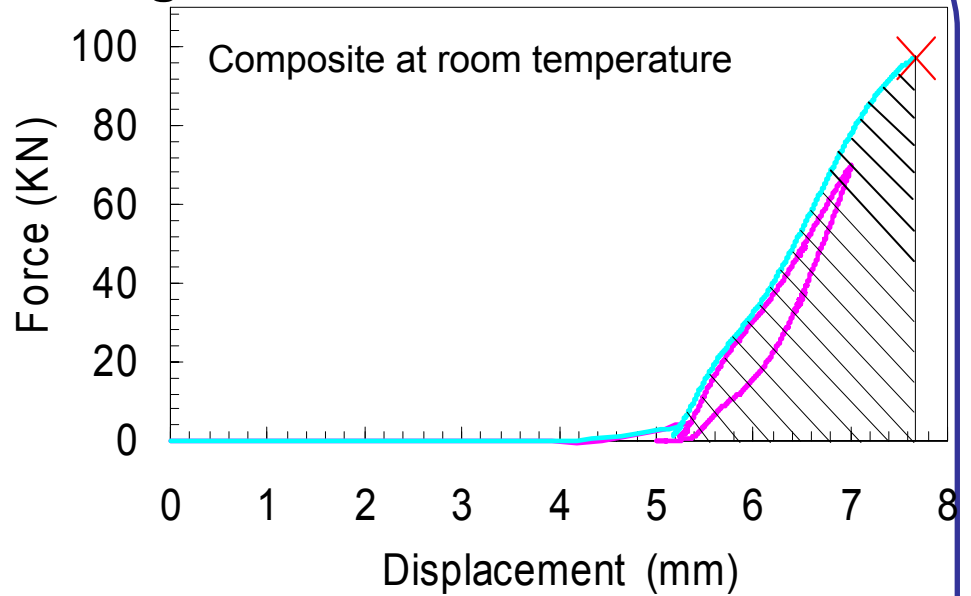
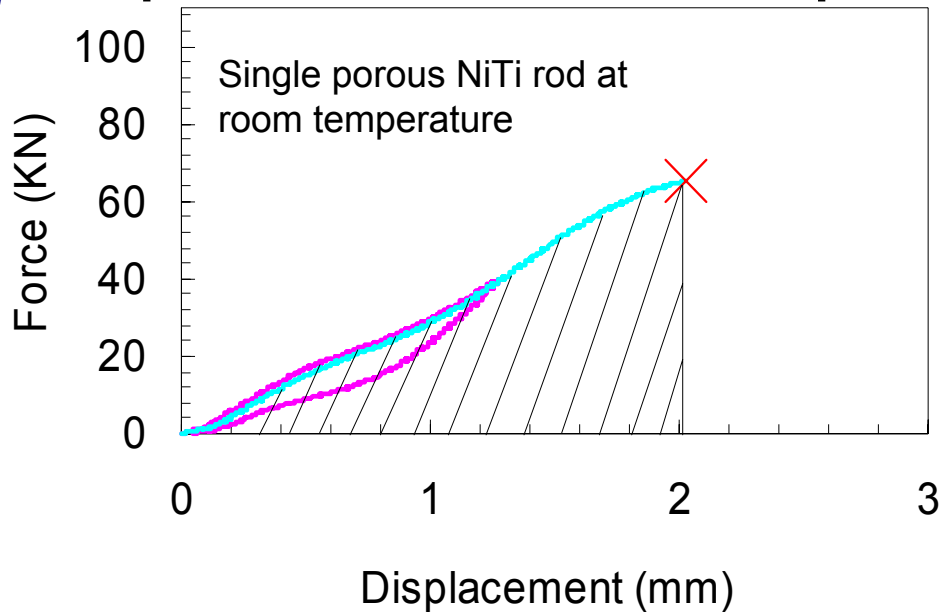
Comparison



Composite Design : Porous NiTi Cylinder and NiTi Spring



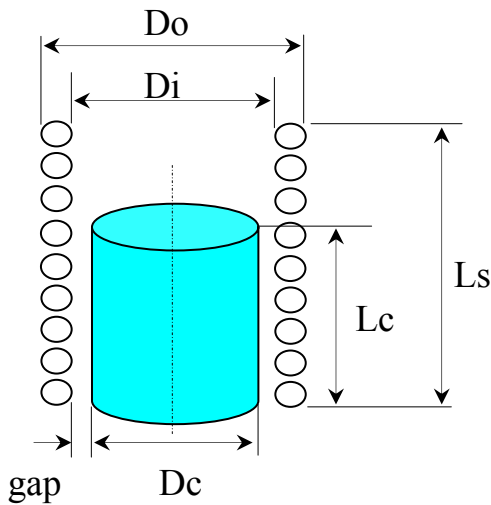
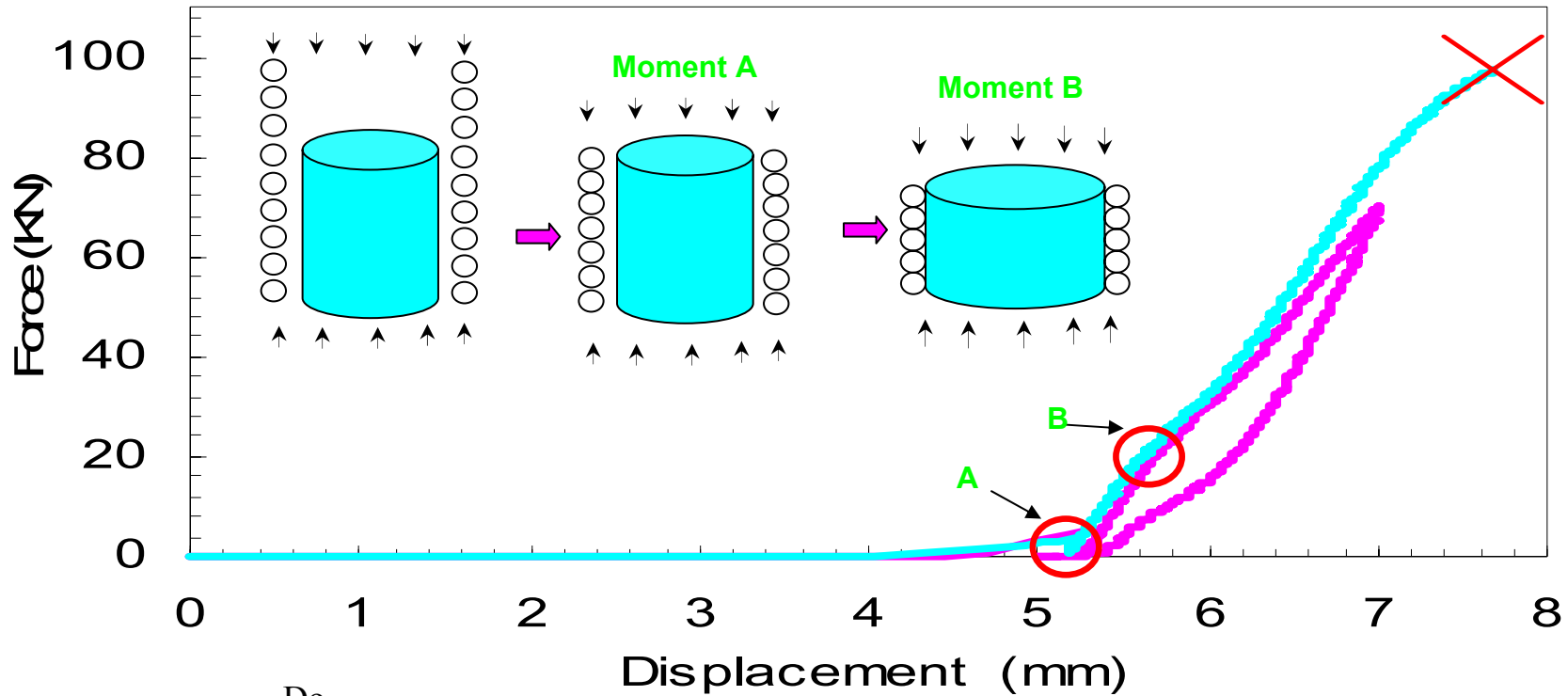
Experimental Results of Composite Design



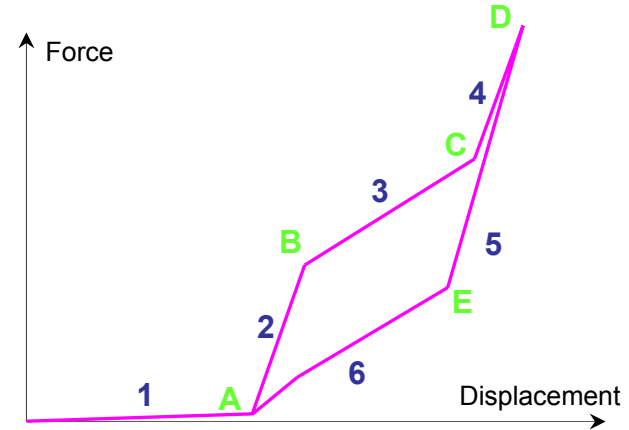
	Maximum Reversible Disp. (mm)	Maximum Reversible Force (KN)	Fracture Disp. (mm)	Fracture Force (KN)	Energy Absorption (EA, J)	Specific EA (J/g)
Single Porous NiTi Rod	1.29 mm	40.74 KN	2.03 mm	65.15 KN	65.1J	12.2 J/g
Composite Structure	1.75 mm	68.76 KN	2.41 mm	97.21 KN	135.5J	15.3 J/g

	Energy Absorbed per Volume (MJ/m ³)
Single Porous NiTi Rod	68.3MJ/m ³
Composite Structure	141.5MJ/m ³
Al Foam	20.0MJ/m ³

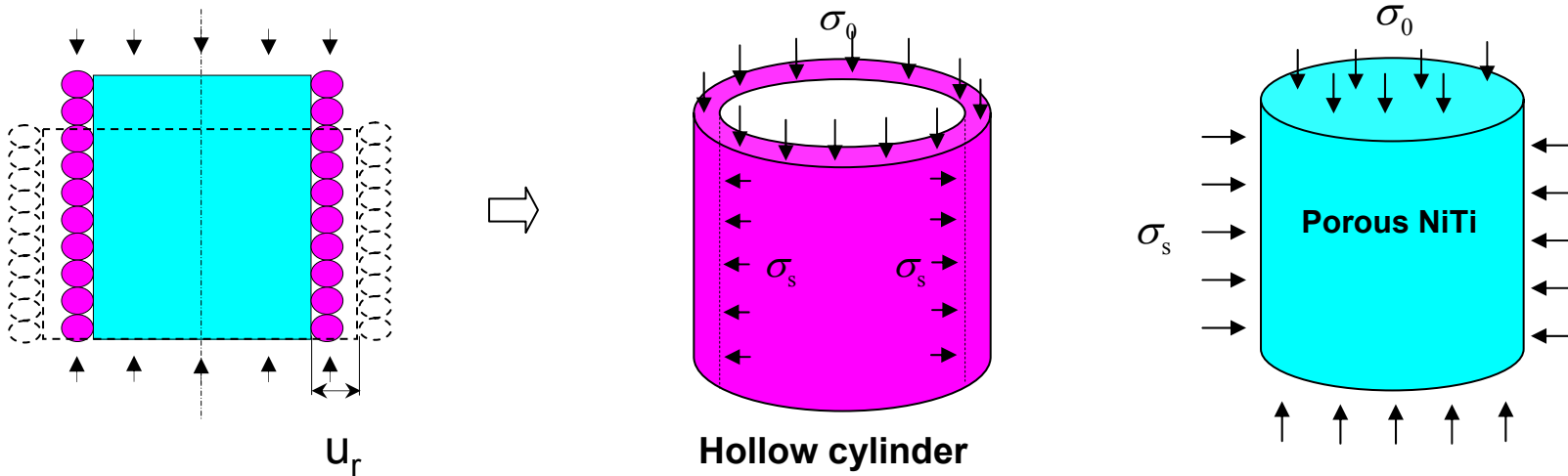
Model for Composite Design



$L_s = 20.3\text{mm}$
 $D_o = 12.4\text{mm}$
 $D_i = 9.1\text{mm}$
 $L_c = 15.0\text{mm}$
 $D_c = 9.0\text{mm}$
 gap = 0.05mm
 $\Delta h = 5.3\text{mm}$



Force-Displacement relation after porous NiTi touches the NiTi spring



Substitute the strain-displacement relation into the stress-strain relation and then substitute into the equilibrium equation

$$\frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \frac{u}{r^2} = 0 \quad \Rightarrow \quad u_r = Ar$$

Boundary Condition - 1

Stress in the solid cylinder $\sigma_r \left(r = \frac{D_c'}{2} \right) = \frac{E_c}{1-\nu^2} A + \frac{\nu}{1-\nu} \sigma_0 = \sigma_s$

Displacement in hollow cylinder at its inner surface

$$u_r^s = \frac{R_i}{E_s (R_o^2 - R_i^2)} \left\{ [(1-2\nu)R_i^2 + (1+\nu)] \sigma_s + \nu \sigma_0 R_o^2 \right\}$$

Displacement in solid cylinder at its surface

$$u_r^c = A \frac{D_c'}{2}$$

Boundary Condition - 2

The radial displacement of the solid cylinder and the inner surface of the hollow cylinder should be equal to keep the continuity of the whole structure

$$u_r^c = u_r^s$$

Solve the above equation for constant A

$$A = \frac{C_1 C_2 \frac{\nu}{1-\nu} + C_1 \nu R_o^2}{C_1 C_2 \frac{E_s}{1-\nu^2} + \frac{D_c'}{2}}$$

$$C_1 = \frac{R_i}{E_s (R_o^2 - R_i^2)}$$

$$C_2 = -\left[(1-2\nu^2)R_i^2 + (1+\nu)R_o^2 \right]$$

Loading Curve

A: The spring shrinks to the same height as of the rod

$$F_A = \frac{G_s d^2}{64nR_o^2} \delta_A$$

B: The rod touches the spring

$$\delta_B = 0.45\text{mm}$$

$$F_B^c = 14\text{KN}$$

$$F_B^s = \pi(R_o^2 - R_i^2)E_s \delta_B / L_c$$

$$F_B = F_B^c + F_B^s$$

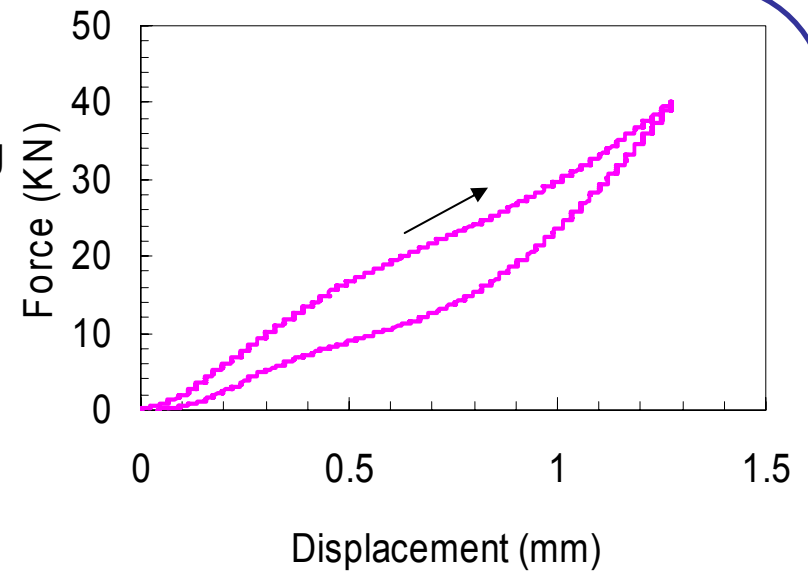
C: Finish martensite transformation

$$\delta_C = 0.92\text{mm}$$

$$F_C^c = \frac{\pi \left(\delta_C - \frac{2\nu}{1-\nu} Ah_C \right) E_T^c (D_C^c)^2}{4h_C (1 - 2\nu^2 / (1-\nu))}$$

$$F_C^s = \frac{\pi \left(\delta_C - \frac{2\nu}{1-\nu} Ah_C \right) E_s (R_o^2 - R_i^2)}{h_C (1 - 2\nu^2 / (1-\nu))}$$

$$F_C = F_C^c + F_C^s$$



D: Maximum reversible point

$$\delta_D = 1.75\text{mm}$$

$$F_D^c = \frac{\pi \left(\delta_D - \frac{2\nu}{1-\nu} Ah_D \right) E_M^c (D_D^c)^2}{4h_D (1 - 2\nu^2 / (1-\nu))}$$

$$F_D^s = \frac{\pi \left(\delta_D - \frac{2\nu}{1-\nu} Ah_D \right) E_s (R_o^2 - R_i^2)}{h_D (1 - 2\nu^2 / (1-\nu))}$$

$$F_D = F_D^c + F_D^s$$

Unloading Curve

E: Start austenite transformation

$$\delta_E = 0.90\text{mm}$$

$$F_E^c = \frac{\pi \left(\delta_E - \frac{2\nu}{1-\nu} Ah_E \right) E_M^c (D_E^c)^2}{4h_E (1-2\nu^2/(1-\nu))}$$

$$F_E^s = \frac{\pi \left(\delta_E - \frac{2\nu}{1-\nu} Ah_E \right) E_s (R_o^2 - R_i^2)}{h_E (1-2\nu^2/(1-\nu))}$$

$$F_E = F_E^c + F_E^s$$

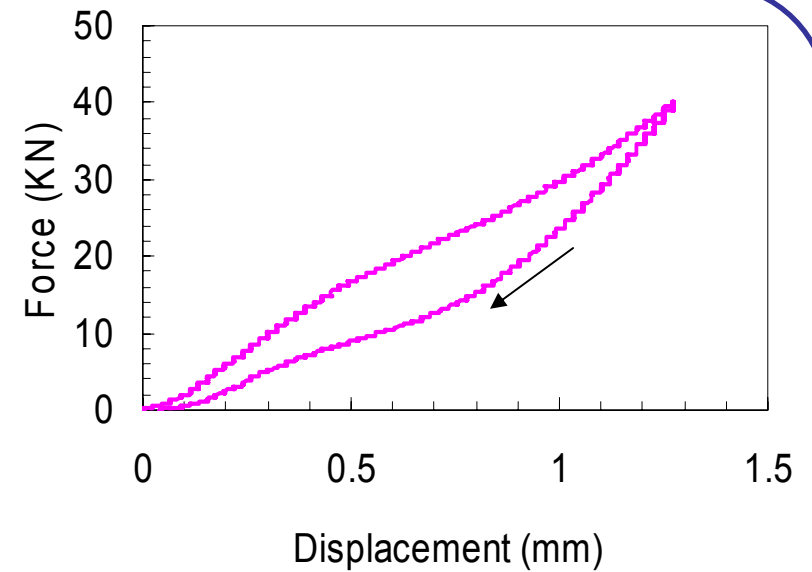
F: Finish austenite transformation

$$\delta_F = 0.35\text{mm}$$

$$F_B^c = 7\text{KN}$$

$$F_F^s = \pi (R_o^2 - R_i^2) E_s \delta_F / L_c$$

$$F_F = F_F^c + F_F^s$$



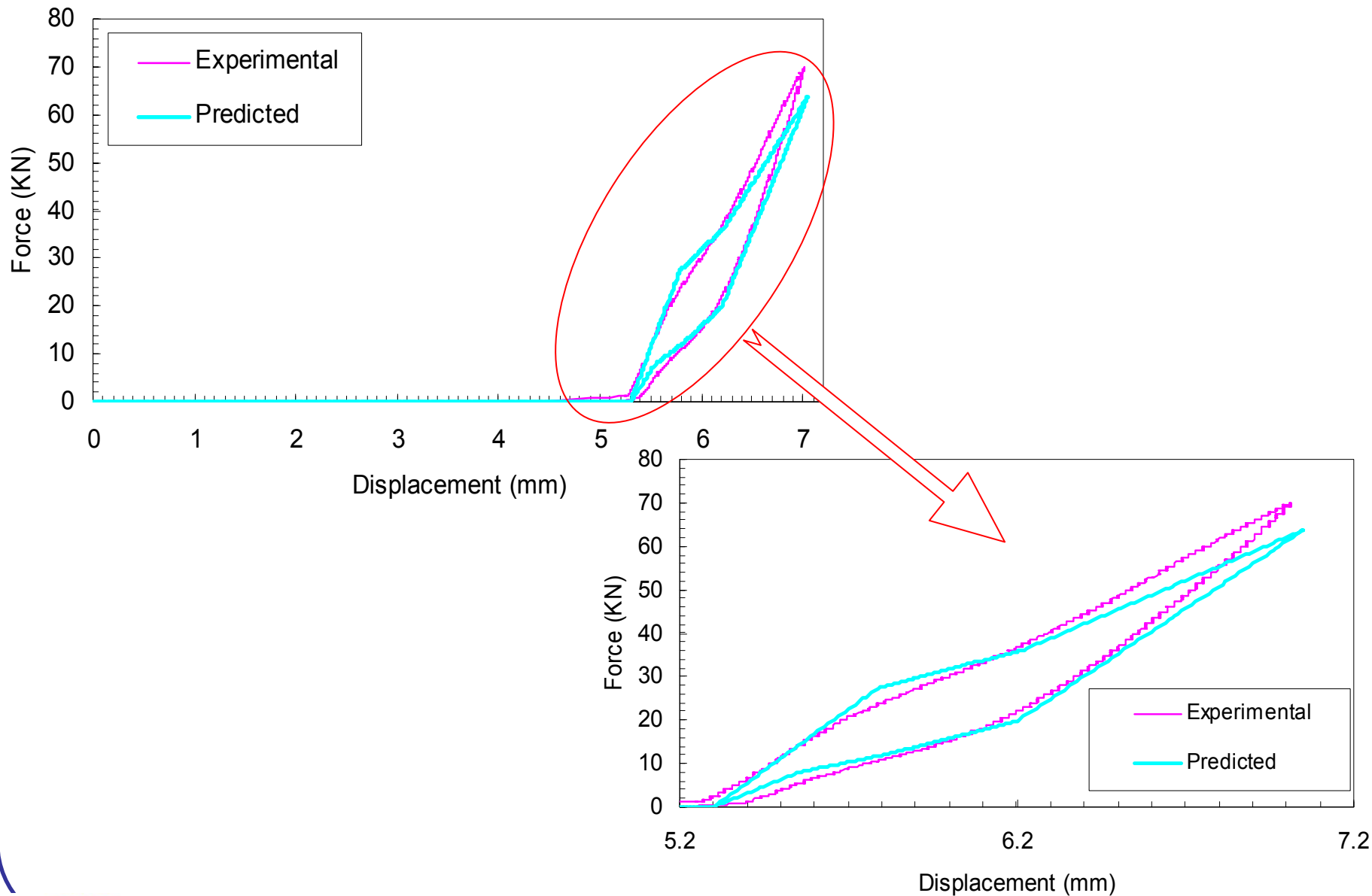
where

$$A = \frac{\left(C_1 C_2 \frac{\nu}{1-\nu} + \nu C_1 R_o^2 \right) \sigma_c^0}{C_1 C_2 \frac{E_M}{1-\nu^2} + \frac{D_B^c}{2}}$$

$$C_1 = \frac{R_i}{E_s (R_o^2 - R_i^2)}$$

$$C_2 = -\left[(1-2\nu^2) R_i^2 + (1+\nu) R_o^2 \right]$$

Comparison of the predicted and experimental Force-Displacement curve



Conclusion

1. Successful processing ductile porous TiNi with high specific energy absorption capability by SPS
2. Establishment of analytical modeling of porous TiNi by micromechanic model
3. Design and demonstration of Composite made of concentric porous TiNi-SE and Spring of TiNi-SE grade