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1 **Laboratory investigation of the impact of lateral spreading on**
2 **buoyancy flux in a river plume**

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ABSTRACT

4
5 We investigate the relationship between lateral spreading and mixing in stratified gravity
6 currents by comparing laterally confined and unconfined currents in a series of laboratory
7 experiments. The vertical turbulent buoyancy flux is determined using a control volume
8 approach with velocity and density fields derived from combined particle image velocime-
9 try (PIV) and planar laser induced fluorescence (PLIF). Lateral spreading is determined
10 in the unconfined experiments based on plan-view imaging using the Optical Thickness
11 Method (OTM). We find that lateral spreading dramatically modifies the plume structure;
12 the spreading plume layer consists of approximately linear density and velocity profiles that
13 extend to the surface, whereas the channelized plume profiles are uniform near the surface.
14 Lateral spreading decreases the average plume density relative to laterally confined currents
15 with similar inflow conditions. However, the local turbulent buoyancy flux in the spreading
16 experiments is approximately equal to that in the confined experiments. This apparent para-
17 dox is resolved when the plume areas are taken into account. The total mixing integrated
18 over the horizontal plume area is significantly higher in the spreading experiments. Thus,
19 the experiments suggest that spreading does not appreciably alter the turbulent mixing pro-
20 cesses at the base of the plume. However, it significantly increases the area over which this
21 mixing occurs and, through this mechanism, increases the net dilution of river water at a
22 fixed distance from the river mouth. Finally, we hypothesize that the spreading does not sig-
23 nificantly increase the local turbulent buoyancy flux because spreading occurs preferentially
24 near the surface, whereas buoyancy flux is greatest in the core of the current.

1. Introduction

Rivers play a critical role in the exchange of material between land and the ocean. The concentration of river-borne matter and the buoyancy of river-influenced coastal waters is determined in large part by the intense mixing that occurs in the initial adjustment of the freshwater as it is discharged into the ocean. During this process outflowing buoyant river water propagates along the ocean surface and expands laterally due to the horizontal baroclinic pressure gradients. Analysis of most field observations relies on model plume propagation and mixing using theory derived from classical two-dimensional laboratory experiments, however, which do not account for the possible effects of lateral spreading. In this contribution we compare the dynamics and mixing in laboratory generated constant flux buoyant gravity currents with and without lateral spreading.

Much of our understanding of gravity current propagation is based on early lock exchange experiments, in which fluids of different densities are initially separated by a vertical barrier and then released suddenly by removing the gate. Based on potential flow theory von Karman (1940) predicted that heavy fluid with density ρ_i propagates into a semi-infinite lighter ambient fluid with density ρ_0 with a mean velocity

$$U_f = \sqrt{2g'h}, \quad (1)$$

where $g' = \frac{\Delta\rho}{\rho_0}g$ is the reduced gravity, g represents gravity, $\Delta\rho$ is the difference between the density of intrusion current and the density of the ambient water, and h is the mean current thickness behind the front. The current front is defined as the narrow region across which density changes abruptly (Garvine and Monk 1974). This was later revisited by Benjamin (1968), who arrived at the same conclusion based on energy-conserving theory for inviscid fluids. More recently Shin et al. (2004) performed experiments on surface gravity currents in a deep ambient fluid, and concluded that the Froude number $Fr = U/\sqrt{g'h}$ approaches 1 in the limit of an infinitely deep environment rather than the larger value of $\sqrt{2}$ predicted

49 by classical work of von Karman (1940) and Benjamin (1968).

50 There are two effects not considered in these classical theories, which are important to
51 the interpretation of river plumes as gravity currents. First, the lock exchange experiments
52 involve a fixed volume of fluid, while the river inflow is fed by constant or tidally varying
53 freshwater discharge from the estuary. Simpson (1997) showed that currents resulting from
54 a lock release and a constant flux currents are different. For example, the frontal speed of
55 a lock-exchange flow decreases with the depth of the active layer while a constant flux flow
56 is independent of the depth of the active layer. Hallworth et al. (1996) concluded that the
57 entrainment into the head of a constant flux gravity current is smaller than in the release
58 of a fixed volume of fluid. They attributed this difference to the continual replenishment
59 of fluid in the head by the constant feed of undiluted fluid from the tail. They compared
60 the entrainment mechanisms and regions that entrain light fluid for constant-volume and
61 constant-flux gravity currents. Unlike a concentrated frontal bore followed by a thin tail in
62 the fixed volume case, the depth of the front and tail were the same in the constant flux case.
63 Kilcher and Nash (2010) described a recent field study on the Columbia River plume and
64 showed that varying flow rates result in significant differences in plume structure, mixing
65 and momentum balance.

66 A second important difference between classical gravity current experiments and river
67 plumes is that most prior experiments use a configuration in which the gravity currents
68 are confined in a straight channel, whereas river plumes discharging into the ocean are
69 laterally unconfined. A number of previous studies use axisymmetric configurations such as
70 sector tanks to incorporate lateral effects (Britter and Simpson 1978; Chen 1980; Didden
71 and Maxworthy 1982; Huppert and Simpson 1980; Patterson et al. 2006), noting significant
72 structural differences in the spreading currents. For example, lateral spreading modifies the
73 frontal propagation speed (Didden and Maxworthy 1982) and vortex stretching associated
74 with lateral spreading gives rise to new vortical structures (Simpson 1997). Patterson et al.
75 (2006) observed that dense fluid propagates at a relatively constant depth in channelized

76 experiments, whereas it appears to be concentrated within the front bore in experiments
77 with a 10 degree sector tank. These experimental results are consistent with Cantero et al.
78 (2007), who showed that Kelvin-Helmholtz (K-H) vortices that formed at the interface in
79 cylindrical lock release currents eventually merged to form a vortex ring near the head. The
80 concentrated vorticity at the head of the cylindrical current initially intensifies due to vortex
81 stretching (Patterson et al. 2006), causing the current to develop a highly turbulent front
82 with a relatively shallow calm body (Cantero et al. 2006). An overview of the modeling of
83 high Reynolds number gravity currents in two-dimensional and axisymmetric configurations
84 is provided by Ungarish and Zemach (2005).

85 We expect that the modification of gravity current structure due to lateral spreading
86 observed in the studies above will be relevant to the interpretation of coastal river plumes.
87 However, the spreading dynamics observed in the previous experiments are somewhat dif-
88 ferent than river inflows because the currents were either forced to spread cylindrically or
89 modified by the tank shape in those cases. At river mouths, the freshwater is initially chan-
90 nelized in the estuary and subsequently begins to spread once it reaches the coastal ocean.
91 During this transition to the unconfined state it undergoes vertical and lateral adjustments,
92 and the spreading rate is set dynamically based in part on the initial momentum and the den-
93 sity of the buoyant layer. Thus, the dynamics of unconfined gravity current generated with
94 a constant freshwater discharge are expected to be different from both the axisymmetric and
95 the sector tank gravity currents. To the authors' knowledge, the laboratory and numerical
96 investigation of freely propagating gravity currents has only been reported by Rocca et al.
97 (2008), who focused on the bottom roughness effects on three-dimensional gravity current
98 propagation.

99 A number of studies have examined the detailed structure of river plumes observed in
100 the field and made direct comparisons with prior results from gravity current experiments.
101 Luketina and Imberger (1987) presented field observations of a tidally pulsed buoyant plume
102 and described in detail an overturning roller at the plume front. Surface water behind the

103 front overtook the roller and formed an energetic mixing area following the front, consistent
 104 with the previous laboratory work from Britter and Simpson (1978). Other field observa-
 105 tions (Wright and Coleman 1971; Hetland and MacDonald 2008) suggested that the lateral
 106 spreading of buoyant plumes should behave like a lateral lock-exchange flow, which means
 107 the propagation in the alongshore direction has a similar speed as the offshore propagation
 108 in the classical lock-exchange experiment. The lateral spreading rate is proportional to the
 109 local internal gravity wave speed, c : $DW/Dt = 2c$, where W is the plume width. Since
 110 the offshore propagation speed of the buoyant layer well behind the front is initially set
 111 by the outflow momentum, this implies that the lateral spreading depends on the inflow
 112 Froude number $Fr_i = U_0/\sqrt{g'_0 H_0}$, where U_0 , g'_0 and H_0 are inflow velocity, reduced gravity
 113 and water depth, respectively. Hetland and MacDonald (2008) and Chen et al. (2009) both
 114 suggested that lateral spreading is significantly affected by mixing in the near-field region.

115 Mixing due to vertical buoyancy flux through the base of the plume is commonly assumed
 116 to occur due to turbulent stratified shear layer processes (MacDonald and Geyer 2004)
 117 including Kelvin-Helmholtz (K-H) instabilities (e.g. Thorpe 1973), though mixing generated
 118 by these processes is still not well-understood (Ivey et al., 2008). Christodoulou (1986)
 119 provided an overview of turbulent mixing at the density interface through theoretical analysis
 120 and re-examination of more than 10 experimental data sets including Ellison and Turner
 121 (1959), Chu and Vanvari (1976), Buch (1980) and Pedersen (1980). He developed a general
 122 law that relates the entrainment rate ($E = w_e/U$) to the bulk Richardson number, $Ri_b = \frac{g'H}{U^2}$.
 123 Here w_e is the vertical, or entrainment, velocity through the reference isopycnal, U is the
 124 characteristic horizontal velocity, and H is the plume thickness. This analysis suggests that
 125 entrainment across the interface occurs in two different regimes. For low bulk Richardson
 126 number ($Ri_b < O(1)$), which corresponds to supercritical flow conditions, Christodoulou
 127 (1986) suggests that "vortex entrainment" occurs, in which vortices at the density interface
 128 actively distort the interface and generate turbulent mixing in the form of Kelvin-Helmholtz
 129 instabilities. In this regime, Christodoulou (1986) finds that $E \approx Ri_b^{-1/2}$. When $Ri_b > O(1)$

130 and the flow is subcritical, mixing is generated by the continuous bombardment of the
 131 interface by vortices generated away from the interface (Linden, 1973) as undulations in the
 132 form of Holmboe waves. This process, which is referred to as "cusp entrainment", results
 133 in a relationship with a larger exponent; $E \approx Ri_b^{-3/2}$. Both mechanisms are active in the
 134 intermediate range when Ri_b is near its critical value of unity.

135 Field measurements of stratified turbulence processes in plumes generally use turbulent
 136 microstructure (Nash et al., 2009) or control volume estimates (MacDonald and Geyer 2004)
 137 to determine ε , the turbulent kinetic energy dissipation rate, or $B = \frac{g}{\rho_0} \overline{\rho'w'}$, the turbulent
 138 buoyancy flux. Following from Ivey and Imberger (1991), MacDonald and Geyer (2004)
 139 used the non-dimensional variable $\frac{\varepsilon}{\Delta u g'}$ to represent the conversion of mean flow energy into
 140 turbulent kinetic energy by shear flow processes, where Δu and g' represent the shear and
 141 stratification, respectively. Subsequently, MacDonald and Chen (2012) introduced the mix-
 142 ing parameter $\xi = \frac{B}{\Delta u g'}$ to represent the non-dimensional turbulent buoyancy flux. Typically,
 143 B is estimated based on the control-volume method, while ε is obtained from microstructure
 144 measurements. The two can be related assuming that $P = \varepsilon + B$, where $P = -\overline{u'w'} \frac{\partial u}{\partial z}$ is
 145 the rate of turbulent kinetic energy production from the mean flow shear. This assumption
 146 is valid in homogeneous and stationary turbulence. B and P may be further related based
 147 on the assumption of constant mixing efficiency for stratified shear mixing associated with
 148 K-H billows. Ivey and Imberger (1991) suggested that K-H billows have an overturn Froude
 149 number $Fr_T = (L_o/L_t)^{2/3}$ close to unity, where $L_o = (\varepsilon/N^3)^{1/2}$ is the Ozmidov scale, L_t is
 150 the representative turbulent length scale and $N^2 = -\frac{g}{\rho_0} \frac{\partial \rho}{\partial z}$ is the buoyancy frequency. For
 151 this mechanism the mixing efficiency $R_f = \frac{B}{P} \simeq 0.2$, and thus $\varepsilon \simeq 4B$.

152 MacDonald and Chen (2012) investigated the relationship between spreading and mixing
 153 using a theoretical model and observations from the Merrimack River plume MA. They
 154 suggest that a quadratic relationship exists between the mixing parameter ξ and a lateral
 155 spreading parameter $\phi = \delta \frac{h}{\Delta u}$, where δ is the strain rate and h is the shear layer thickness.
 156 Their theoretical model is predicated on the idea that lateral spreading stretches individual

157 K-H billows that are oriented transverse to the mean flow direction, and hence increase the
158 intensity of the billows due to conservation of angular momentum. The model predicts that
159 the turbulent dissipation rate is proportional to the square of total plume width, thus linking
160 turbulent mixing and plume spreading. They show that estimates of ξ and ϕ based on data
161 from the Merrimack River plume agree with their theoretically derived relationship. The
162 evidence from their study provides strong support for the hypothesis that spreading increases
163 mixing. However, the conclusions from the field experiment are necessarily correlative, since
164 it was not possible to control the spreading rate externally. The objective of the laboratory
165 study described herein is to isolate the effect of spreading on mixing in the plume so that
166 the relationship between the two can be clearly analyzed. We do this by conducting two
167 identical sets of experiments, one in which the plume is confined between channel walls and
168 is not allowed to spread, and one in which it is allowed to spread freely.

169 The manuscript is organized as follows. In Section 2, we describe the confined and uncon-
170 fined experimental configurations and the two main measurement approaches. We present
171 the experimental results in Section 3. In Section 4 we describe the observed relationship
172 between spreading and mixing, and present possible hypotheses to explain the relationship.
173 Our conclusions are summarized in Section 6.

174 **2. Experimental Set-up**

175 A schematic of the laboratory set-up is shown in Figure 1. All experiments were con-
176 ducted in a water tank (hereafter called the plume basin), which is 400 cm long, 250 cm
177 wide and 50 cm deep. The buoyant water source was a 600 l constant head tank located 5 m
178 above the level of the plume basin. The inflow was introduced into the plume basin through
179 a small estuary tank, containing a diffuser board and a honeycomb to achieve uniform flow
180 and a 30 cm (W_0) \times 5 cm (H_0) channel section. The origin of the Cartesian coordinate is
181 defined at the center of the river mouth water surface, x is the onshore coordinate, y is the

182 alongshore coordinate, and z is the vertical coordinate with positive upward. The coordi-
183 nate system and the plume basin dimensions are shown in the schematics in Figure 1. Each
184 experiment started by opening the estuary gate and the buoyant water valve simultaneously.
185 After propagating across the plume basin, mixed fluid exited the system over an adjustable
186 weir at the downstream end of the basin.

187 Two configurations were designed with different estuary tank locations to determine
188 how lateral spreading affects plume structure. In the unconfined case the estuary tank was
189 located at the center of one end of the plume basin (Figure 1a). By lifting the gate and
190 opening the valve, the buoyant fluid was released from the estuary channel into the ambient
191 water where it was allowed to spread freely. For the confined case the estuary tank was
192 oriented between the tank wall and a vertical plastic wall so that these formed transparent
193 lateral boundaries (Figure 1b). The current stayed within the channel as in the classical
194 lock-exchange experiment, but received a constant flux of buoyant fluid.

195 *a. Plume structure and width measurements*

196 For visualization purposes, the freshwater was dyed with colored food dye and the plume
197 freshwater thickness field was measured using the optical thickness method (OTM) (Cenedese
198 and Dalziel 1998; Yuan et al. 2011). The food dye was added in the source water and
199 illuminated by a point light source located above the plume basin. A sequence of images
200 were acquired with a digital camera mounted perpendicular to the water surface. We describe
201 the plume based on the freshwater thickness h_e , which is calculated according to:

$$\frac{I(h_e, C_0)}{I_0(h_e, 0)} = e^{-\theta C_0 h_e}, \quad (2)$$

202 where $I(h_e, C_0)$ is the transmitted intensity of light passing through a distance h_e of fluid
203 with dye concentration C_0 . The attenuation coefficient θ is determined using a calibration
204 with a wedge-shape cuvette before each experiment. The effects of absorbed light by the

205 tank bottom, as well as the non-uniformity of the background light from a single point source
206 are minimized by normalizing the intensity of dyed water to the transmitted intensity for
207 the same thickness of undyed water, $I_0(h_e, 0)$, at each pixel.

208 Note that h_e is the 'effective' freshwater thickness, which is equivalent to the unmixed
209 plume layer thickness (Yuan et al. 2011). The real thickness of dyed water in the plume is
210 always larger than the effective freshwater thickness we measured, as the plume is contin-
211 ually diluted by entraining undyed ambient salt water. However, these experiments, which
212 we refer to as the plan-view experiments (Figure 1c), provided comprehensive imaging of the
213 entire plume structure that compliments the detailed quantitative velocity and density mea-
214 surements described in §2.b. The data from the plan-view experiments are used to determine
215 the plume width and spreading rate.

216 *b. Velocity and density measurements*

217 We investigated the detailed interfacial dynamics and mixing processes in the plume
218 (Figure 1d) using a combined particle image velocimetry (PIV) and planar laser induced
219 fluorescence (PLIF) technique developed by Cowen et al. (2001). A description of this
220 method and its implementation for stratified flows using a similar set-up are given by Horner-
221 Devine (2006). This technique measures velocity and density fields at short time intervals
222 from a sequence of image triplets taken with a digital camera fitted with a wavelength cut-off
223 filter. It requires laser sheets from both ND:YAG laser (Solo 200XT, New Wave Laser) and
224 Argon ion laser (Innova 306, Coherent) located beside the plume basin. The laser beams are
225 directed horizontally beneath the plume basin and steered vertically through a cylindrical
226 lens to produce vertical laser sheets that are located on the plume axis and are carefully
227 aligned in the offshore direction. A 1024 by 1024 pixel CCD camera (Dalsa Cooperation) is
228 positioned 1 m from the plume centerline, and provides images in the vertical - offshore plane
229 of the plume with a 12 cm by 14 cm field of view. The velocity field is obtained from the first
230 two images within one triplet sequence. These images are illuminated by the ND:YAG laser,

231 which has a wavelength higher than the cut-off wavelength so that the light scattered from
232 the particles can pass the filter. The images are processed to generate velocity fields using
233 matPIV (Sveen 2004). The third image in the sequence is illuminated by the Argon ion laser
234 and processed using the PLIF technique (Crimaldi 2008). The PLIF concentration images are
235 converted to density using a MicroScale Conductivity and Temperature Instrument (MSCTI,
236 PME Inc.) probe located at the outer edge of laser field. The probe was mounted to a vertical
237 profiler generated by a step motor and controlled by an Arduino board.

238 Ten plan-view experiments were conducted in the spreading configuration prior to the
239 quantitative PIV-PLIF experiment. Sixteen runs with PIV-PLIF method were conducted in
240 total, with eight in the channelized configuration and eight in the spreading configuration.
241 Each set of channelized and spreading experiments were designed to have similar inflow
242 conditions (similar Fr_i). The inflow condition was controlled by the inflow flow rate Q_0
243 and inflow reduced gravity $g'_0 = \frac{\rho_0 - \rho_i}{\rho_0} g$, where ρ_0 is the ambient water density and ρ_i is the
244 inflow water density. However, due to experimental limitations, it was impossible to match
245 g'_0 exactly. Experimental parameters for all runs are provided in Table 1. The inflow Froude
246 number (Fr_i) ranged from 0.25 to 2.14, including subcritical and supercritical conditions.
247 The inflow Reynolds number ($Re = \frac{\rho_i U_0 H_0}{\mu}$) are listed in the last column, where ρ_i , U_0 and
248 H_0 are the inflow density, velocity and water height, μ is the dynamic viscosity of the water.

249 3. Results

250 a. *General Plume Description*

251 The development of a freely spreading gravity current is shown in the first row of Figures
252 2 (a-c) and 3 (a-c), which show freshwater thickness fields of the flow. Corresponding vertical
253 density fields overlain with simultaneous velocity profiles from the vertical PIV-PLIF mea-
254 surement are shown below the plan-view images (Figures 2d-f and 3d-f). For comparison,
255 vertical density fields and velocity profiles for confined cases with similar inflow conditions

256 are plotted in the third row in Figures 2 (g-i) and 3 (g-i) at the same time.

257 In the unconfined case the inflowing buoyant water is observed to spread laterally and
258 form a cone-shaped surface layer. The structure and evolution of this surface layer depends
259 on the inflow Froude number. In the high inflow Froude number run (PL4: $Fr_i = 1.68$),
260 the flow is supercritical and the plume consists of a jet-like current with an offshore velocity
261 higher than the alongshore velocity (Figure 2a-c). In the low inflow Froude number case
262 (PL6: $Fr_i = 0.50$) the flow is subcritical and the plume shape is semi-circular (Figure 3a-c).

263 There are several points worth noting regarding lateral spreading. First, although the
264 plume shoals significantly in unconfined cases, the frontal bore has a thickness similar to
265 the confined case. The frontal bore is a cavity filled with the lighter fluid, which has the
266 sharp density gradient at the leading edge and the enhanced turbulence at the lee side. This
267 structure is similar to the laboratory observations of lock-exchange experiment (Patterson
268 et al. 2006; Simpson 1997) and to the field observation of small buoyant plumes (Luketina
269 and Imberger 1987; Garvine and Monk 1974). The thick layer in the frontal bore appears
270 as a slightly brighter band, relative to the region immediately inside of it, pointed by the
271 leftmost solid arrow in the freshwater thickness field (Figures 2a-b and 3a-b). In the high
272 Fr_i confined case (Figure 2i), we observe an approximately 6 cm thick frontal bore and a
273 constant thickness (approximately 4 cm) trailing current behind it. In contrast, in the high
274 Fr_i unconfined case (Figure 2f) an approximately 6 cm deep frontal bore is followed by less
275 than 2 cm layer (not shown in the figure). This phenomenon is consistent with observations
276 in previous laboratory studies (Patterson et al. 2006) and numerical simulations (Cantero
277 et al. 2007) on cylindrical spreading gravity currents.

278 In the unconfined cases the density field shows a clear shear-induced vortex billow in the
279 frontal bore (Figures 2e and 3e), which is generated by Kelvin-Helmholtz instability at the
280 interface. The instability is more developed in the unconfined case than the confined case,
281 where the billows are not as distinct. After the front has passed, the interface is continually
282 deformed with a similar structure. On the leading edge of each wave the interface becomes

283 very sharp and the buoyant layer thickens (Figures 2f and 3f). The trailing edge is marked by
284 intense mixing as the buoyant layer thins again, often nearly to zero thickness. This periodic
285 oscillation of the plume thickness is also observed in the plan-view freshwater thickness field
286 as deep bands near the river mouth. This structure, together with the frontal bore structure
287 discussed in the previous paragraph, is marked with arrows in the plan-view freshwater
288 thickness field (Figures 2c and 3c). Although both structures are clearly observed in the
289 freshwater thickness video, they are a little hard to be identified in still images.

290 In contrast, mixing in the confined current occurs within a fairly uniform mixing layer at
291 the interface (Figures 2i and 3i). The interface is more diffuse and the scale of the interface
292 excursions in the mixing region is significantly smaller than in the unconfined case. In
293 particular, there is no evidence of the low-wavenumber interface oscillation structures along
294 the interface. The cause of these structures and their relationship to plume spreading will
295 be investigated in future work.

296 The velocity profiles in Figures 2 and 3 are the instantaneous front-relative offshore
297 velocity, generated by subtracting the frontal propagation speed, u_f . The frontal propagation
298 speed is calculated based on a linear fit of front position (x_f) versus time (t). Negative
299 velocities imply that the velocity in the buoyant layer exceeds the frontal propagation speed
300 (i.e., $u > u_f$). These results clearly show that fluid in the plume body is overtaking the
301 plume front (Figures 2e, h and 3e, h). The maximum velocity of the fluid within the plume
302 front and body was typically around 40% to 60% faster than the frontal propagation speed
303 in our experiments. This result is in agreement with earlier numerical simulations (Hartel
304 et al. 2000) and laboratory observations (Thomas et al. 2003) of a two-dimensional gravity
305 current, which find that velocities just behind the gravity current head were typically 20%
306 higher than the frontal velocity. The axisymmetric gravity current was reported to have
307 velocities in the tail up to 40% of that of the front (Patterson et al. 2006).

308 *b. Plume Width and Global Spreading Rate*

309 After sufficient time, the plume is considered to be in a steady state because the flow
310 rate of plume water leaving the basin over the weir matches the inflow discharge. This was
311 confirmed based on the vertical velocity and density fields. The mean plume freshwater
312 thickness, which is based on an average over 100 seconds (500 images), is highest near the
313 plume center and decreases both in the offshore and in the alongshore direction (Figure 4a).
314 The freshwater thickness contours in the high Fr_i run are half ellipses with major semi-axes
315 located at the plume centerline. The freshwater thickness contours in the low Fr_i run are
316 approximately semi-circles (not shown).

317 As discussed in §2, the freshwater thickness h_e is equivalent to the layer thickness in the
318 absence of mixing. It has no dynamical meaning; however, it is valuable for estimating the
319 plume width and the lateral spreading rate since it represents the total amount of freshwater
320 at each point. A centered non-normalized Gaussian fit was successful in describing the
321 lateral distribution of h_e at each location along the plume axis (Figure 4b). We define
322 the plume width in terms of the variance of the Gaussian fit $b = C\sigma$. For the present
323 experiments the coefficient C is set to be 4 for two reasons. First, it is observed to account
324 for most of the cross-sectional area in the Gaussian distribution. Second, this value fits the
325 freshwater conservation (the detailed calculation will be discussed in §3.e). Moving offshore,
326 h_e decreases exponentially (Figure 4c) and the width increases exponentially (Figure 4d).
327 The plume width exponential fit is referenced to a virtual origin at $x = x_0$, which is different
328 for each experiment, but constant within each experiment after the plume reaches steady-
329 state. The corrected offshore distance, r , is defined as the distance from the virtual origin
330 $r = x - x_0$. Note that this is different from the virtual origin used to describe frontal
331 propagation (Luketina and Imberger 1987; Kilcher and Nash 2010). In the case of the front,
332 the virtual origin is defined as the origin of a circular fit to the plume front, which typically
333 moves offshore at a constant speed. In the present experiment, the plume is modeled as
334 expanding radially and so the terminology is consistent with this model.

335 Hetland and MacDonald (2008) defined the global spreading rate as $\alpha = \frac{\Delta r/r}{\Delta b/b}$, which
 336 relates the relative change in width to the relative change in radial distance. The radial
 337 distance is the same as the corrected offshore distance in the present experiment. They
 338 use this parameter to differentiate between divergent plumes ($\alpha < 1$), in which the lateral
 339 expansion of the flow is faster than the offshore propagation, and convergent plumes ($\alpha > 1$)
 340 in which the lateral expansion is slower than the offshore propagation. We determine the
 341 spreading rate by fitting an exponential curve $\sigma = a(x - x_0)^n$ to the width (recall that
 342 $b = 4 \times \sigma$) profile for every case. A simple derivation shows that the global spreading rate,
 343 α , is the reciprocal of the exponent n :

$$\frac{db}{b} = n \frac{d(x - x_0)}{x - x_0} = \frac{1}{\alpha} \frac{dr}{r}. \quad (3)$$

344 The spreading rate α decreases with increasing Fr_i (Figure 5), indicating a shift toward
 345 divergent plumes (e.g. Figure 2a-c) for higher Fr_i . A divergent plume is more like an
 346 energetic jet, the lateral expansion of the flow is faster than the movement of that parcel
 347 away from the source as illustrated in Figure 2b and 2c. A divergent plume has streamlines
 348 that are more splayed than they would be with uniform radial spreading. Lower Fr_i has a
 349 higher spreading rate α , the plume will spread laterally more slowly as it flows away from
 350 the estuary mouth (Figure 3a-c). In the convergent plume case inflow momentum is less
 351 important, radial and lateral spreading are balanced, and the depth contours are circular.
 352 It is important to note that the inflow parameter in plan-view experiments and PIV-PLIF
 353 experiments are not perfectly matched. The spreading rate α of three spreading runs, which
 354 do not have plan-view freshwater thickness field, was extrapolated from Figure 5 and then
 355 was used to calculate the plume width in the control-volume method (§3.e).

356 *c. Density and Velocity Profiles*

357 In Figure 6 the average plume density and velocity profiles are plotted for all sixteen runs
358 with different inflow Froude numbers. The profiles are averaged in time over 500 seconds
359 during the steady state period and in the offshore direction over the PIV/PLIF field of view.
360 The vertical axis is normalized by the plume thickness, H_p , which is defined as the depth
361 of the 95% total freshwater flux contour. Here, the freshwater flux is calculated from the
362 density and velocity field according to $Q_f(x, z) = \int_z^0 \frac{\rho_0 - \rho(x, z)}{\rho_0 - \rho_i} u(x, z) dz$, where u , ρ are the
363 time averaged velocity and density along plume centerline ($y = 0$) over 500s, and hence Q_f
364 is the flux per unit width along the centerline.

365 Density and the velocity profiles are noticeably different in the channelized and spreading
366 cases. In all of the channelized runs, both density and velocity profiles exhibit a step-wise
367 structure consisting of a mixing layer, where the profiles vary nearly linearly, in between a
368 near-surface uniform density layer and the underlying quiescent ambient water. This struc-
369 ture is similar to the density and velocity profiles in channelized gravity current laboratory
370 experiments (Britter and Simpson 1978; Didden and Maxworthy 1982) and in the two-layer
371 flow in salt-wedge estuary channels (MacDonald and Horner-Devine 2008; Tedford et al.
372 2009a). The near-surface uniform density layer disappears in the spreading cases; velocity
373 increases (density decreases) approximately linearly all the way to the water surface. This
374 structure is consistent with field observations from river plumes, including the Columbia
375 (Kilcher et al. 2012), Fraser (MacDonald and Geyer 2004), and Merrimack (MacDonald
376 et al. 2007) river plumes.

377 Profiles from two low Froude number spreading runs (SP5 and SP6, Table 1) have slightly
378 different structures from the rest of the runs. Firstly, the interface thickness for velocity
379 profile is much greater. Secondly, the center of the interface is offset downward in the
380 velocity profile compared with the density profile. Thorpe (1971) suggested that the co-
381 location of the density gradient and velocity shear is essential for generating K-H instabilities.
382 Lawrence et al. (1991) showed that Holmboe instabilities can occur when the shear interface

383 thickness is larger than the density interface and the inflection points of the two profiles are
 384 displaced. This suggests that these two runs may be susceptible to Holmboe instabilities.
 385 Instabilities observed at the interface for run SP5 (Figure 3f) have a similar character to the
 386 Holmboe instabilities observed in exchange flow laboratory experiments by Tedford et al.
 387 (2009b). These low Fr_i instabilities may indicate that the plume is in a slightly different
 388 mixing regime for these two runs. In particular, these runs may exhibit cusp entrainment as
 389 opposed to vortex entrainment as described by Christodoulou (1986).

390 *d. Plume density*

391 The average density of the plume layer at a point sufficiently far from the river mouth
 392 determines the buoyancy available to drive far field plume processes such as alongshore
 393 penetration and freshwater flux, and is related to the net dilution of river-borne matter.
 394 The average density at this point is a consequence of the mixing and advection processes in
 395 the near-field region. Here we compare the average plume density at the offshore end of our
 396 measurement region to see if there is a difference between spreading and channelized plumes.
 397 We define the buoyancy anomaly β as:

$$\beta = \frac{\Delta\rho_0}{\Delta\rho} = \frac{\rho_0 - \rho_i}{\rho_0 - \rho_p}, \quad (4)$$

398 where $\rho_p = \frac{1}{H_p} \int_{-H_p}^0 \rho(z) dz$ is the plume density averaged vertically over the entire plume
 399 layer at plume centerline. The buoyancy anomaly β is the reciprocal of the normalized
 400 density anomaly, $\frac{\Delta\rho}{\Delta\rho_0}$, which quantifies the net mixing (or, plume dilution) that has occurred
 401 between the river mouth and the measurement location (Hetland 2010). Larger values of β
 402 indicate a higher average plume density at the measurement location, thus more mixing has
 403 occurred. The relationship between β and Fr_i is shown in Figure 7. Errors are estimated
 404 based on the standard deviation of the average plume density. The average error for all
 405 spreading and channelized cases are shown separately in Figure 7. We observe that β is

406 higher in the spreading cases than the channelized cases: the average value of β in the
 407 channelized cases is 2.8 ± 0.4 , while in the spreading cases is 3.7 ± 0.6 . Note however that
 408 three points, highlighted by cross symbols in Figure 7, have opposite result. Two low Fr_i
 409 spreading runs (SP5 and SP6) have extremely low value of β . These may be because they
 410 are in a different mixing regime as we discussed in the previous section. The measurement
 411 error associated with low Fr_i spreading runs is also higher than the other runs because of
 412 the limited amount of usable data in PIV measurements. The anomalous result of the low
 413 Fr_i channelized run (CH1) is unclear. The buoyancy anomaly β shows little dependence on
 414 Fr_i in the channelized cases over the parameter range of these experiments.

415 We expect that the buoyancy anomaly observed in the channelized runs is strongly in-
 416 fluenced by the near-surface uniform density layer, which is not actively mixing due to the
 417 lack of density gradient but is included in the calculation of the average density anomaly.
 418 We investigate whether the difference in β between the spreading and channelized runs may
 419 be attributed to the disappearance of this surface layer in the spreading cases, rather than
 420 differences in the intensity of mixing processes at the interface. In order to test this we limit
 421 the averaging in ρ_p to include only the mixing layer. The plume buoyancy anomaly with
 422 only the mixing layer β_{ML} is the same in the spreading (3.7 ± 0.6) and channelized (3.7 ± 0.9)
 423 runs, within the experimental errors. Thus, the modification of the density in the mixing
 424 layers is the same in both cases, suggesting that the intensity of mixing is the same. This
 425 result suggests that the differences observed in the buoyancy anomaly β (Figure 7), must be
 426 due to advection processes rather than mixing. Differences in mixing will be quantified in
 427 terms of the entrainment rate and buoyancy flux in §3.e and §3.f.

428 Based on a simple theoretical model, Hetland (2010) predicts that the normalized density
 429 anomaly anomaly ($1/\beta$) at the end of near-field plume is a function of $w_e b_0^2 / Q_{f0}$, where w_e
 430 is the entrainment velocity, and b_0 and Q_{f0} are the inflow width and freshwater flow rate,
 431 respectively. The field of view in our experiment is close to the end of the near-field region,
 432 though it is likely to be somewhat inside the near-field due to constraints in the experimental

433 set-up. The buoyancy anomaly at this point may provide insight into how $\Delta\rho$ changes with
434 inflow conditions and lateral boundary conditions. In the Hetland (2010) model, increasing
435 freshwater inflow results in a decreasing normalized density anomaly at the end of near field
436 region. The decrease in the normalized density anomaly, which appears as an increase in β ,
437 with Fr_i observed in the spreading cases (Figure 7 black dash line) is consistent with this
438 prediction. Leaving out two anomalous points (SP5 and SP6), we still observe an increase in
439 β with Fr_i (Figure 7 shaded dash line) in spreading runs but with a significant smaller slope.
440 On the other hand, the tendency of β on Fr_i in the channelized runs (Figure 7 black and
441 shaded solid lines) are opposite with and without the anomalous run (CH1). However, both
442 slopes are small compared to spreading ones thus we conclude that no clear relationship is
443 observed between β and Fr_i in the channelized runs.

444 *e. Total Vertical Density Flux and Entrainment Velocity*

445 A direct measurement of the bulk entrainment into the freshwater plume is obtained by
446 calculating the total vertical density flux through the plume base using a control volume
447 approach (MacDonald and Geyer 2004). This technique has been successfully applied for
448 measuring buoyancy, momentum and sediment fluxes in the Merrimack (MacDonald et al.
449 2007) and Columbia (Kilcher et al. 2012; Nowacki et al. 2012) river plume. MacDonald
450 et al. (2007) confirmed that the control volume results have an excellent agreement with
451 the Regional Ocean Modeling system (ROMS) numerical model output, and are consistent
452 with direct measurement of turbulent dissipation by autonomous underwater vehicle (AUV)
453 microstructure. The method involves conservation of volume and mass within a control
454 volume (CV) bounded by the following control surfaces (CS): river mouth, end of the field of
455 view, two lateral boundaries, water surface and a specified bottom isopycnal. Conservation
456 of volume and mass are expressed according to

$$\iint_{CS} \vec{U} \cdot d\vec{A} = \frac{\partial}{\partial t} \left[\iiint_{CV} dV \right]; \quad (5)$$

$$\iint_{CS} \rho \vec{U} \cdot d\vec{A} = \frac{\partial}{\partial t} \left[\iiint_{CV} \rho dV \right]. \quad (6)$$

457 The right hand side of both equations reduces to zero once the plume layer reaches the
 458 steady state. We assume that there is no flux through the water surface and two lateral
 459 boundaries, i.e., almost all mixing occurs at the plume base. With these simplifications,
 460 equations 5 and 6 equate diapycnal volume and density fluxes with vertically integrated
 461 offshore volume and density fluxes above a specified isopycnal:

$$\int_z^0 \int_{-b/2}^{b/2} u dy dz \Big|_{x=0}^{x=x} - \int_0^x \int_{-b/2}^{b/2} w dy dx = 0; \quad (7)$$

$$\int_z^0 \int_{-b/2}^{b/2} \rho u dy dz \Big|_{x=0}^{x=x} - \int_0^x \int_{-b/2}^{b/2} \rho w dy dx = 0. \quad (8)$$

462 Horizontal volume and density fluxes along the plume centerline are calculated from the
 463 time averaged density (ρ) and offshore velocity (u) profiles shown in Figure 6. The velocity
 464 and density fields are then assumed to be laterally uniform and the integral in the y direction
 465 is accounted for by multiplying by the width, b . There are two different ways to estimate
 466 the plume width. One is from the freshwater conservation in the control-volume analysis
 467 in MacDonald and Geyer (2004): $Q_{f0} = b(x) \int_{-H}^0 \frac{\rho_0 - \rho(x,z)}{\rho_0 - \rho_i} u(x,z) dz$, where H is the total
 468 water depth. The other way to compute the plume width is from the plan-view experiment
 469 (i.e. $b = 4 \times \sigma$) as shown in §3.b. For our experiments, the value of b computed using
 470 freshwater conservation agreed well with the value from plan-view experiments method, and
 471 the latter definition was used in the calculations because it was less noisy. It is important to
 472 note, however, the way to calculate b accounts to some extent for non-uniformity in u and
 473 ρ because of the conservation of freshwater. Equations 7 and 8 can then be re-expressed in
 474 terms of the laterally averaged diapycnal velocity \bar{w} and density flux $Q_v = \frac{g}{\rho_0} \bar{\rho} \bar{w}$ through
 475 each isopycnal,

$$\frac{\partial}{\partial x} \left[\int_z^0 budz \right] - \bar{w}b = 0; \quad (9)$$

$$\frac{\partial}{\partial x} \left[\int_z^0 \rho budz \right] - \bar{\rho} \bar{w}b = 0. \quad (10)$$

476 Finally, the Reynolds salt flux is determined according to $\overline{\rho'w'} = \bar{\rho}\bar{w} - \bar{\rho}\bar{w}$. The vertical
 477 advective density flux $\bar{\rho}\bar{w}$ is calculated by multiplying the diapycnal velocity determined
 478 from equation 9 with the density of the bounding isopycnal. Profiles of \bar{w} show that it is
 479 negative at the surface and positive below the plume base (Figure 8a, c). This is consistent
 480 with the entrainment velocity profile from the Fraser River plume lift-off (MacDonald and
 481 Geyer 2004), indicating a developing mixing layer that is entraining fluid from both the
 482 surface and deep water.

483 Channelized and spreading cases both show similar structure in the total vertical density
 484 flux and entrainment velocity profiles (Figure 8). Note that in the channelized cases the
 485 maximum positive and the maximum negative entrainment velocity are almost the same,
 486 while in the spreading cases the maximum positive entrainment velocity from below is larger
 487 than the maximum negative entrainment velocity at the surface. This is because there
 488 is a distinct, uniform upper layer in the channelized case from which the buoyant fluid
 489 can be entrained down to the mixing layer. In the spreading case, however, there is no
 490 such freshwater source at the surface so the entrainment into the mixing layer from above
 491 (negative \bar{w}) is smaller than from below (positive \bar{w}).

492 For comparison with previous experiments we compute the entrainment rate, defined
 493 as $E = w_e/U$, where w_e is the maximum vertical velocity through the lowest isopycnal
 494 (i.e., the maximum value of \bar{w} in the profile) and U is the layer-averaged velocity in the
 495 plume. Morton et al. (1956) first developed the idea of an entrainment rate to quantify
 496 the flow of ambient fluid into the turbulent layer. Ellison and Turner (1959) carried out
 497 surface jet experiments in a long, narrow, rectangular channel, similar to our channelized
 498 configurations. They calculated E based on conservation of volume and related it to the local

499 bulk Richardson number Ri_b . They concluded that E decays exponentially according to Ri_b^γ
500 and that entrainment becomes negligible for $Ri_b > 0.8$ in the surface jet. Christodoulou
501 (1986) summarized all available experimental results for a variety of flow types and proposed
502 governing laws for the dependence of the E on Ri_b . He found that $\gamma \approx -1/2$ at small Ri_b
503 and progressively increases to $-3/2$ for large Ri_b in buoyant overflows.

504 We calculate the local bulk Richardson number using $Ri_b = \frac{g'_m H_p}{U_m^2}$, where g'_m is the
505 maximum reduced gravity corresponding to the maximum density anomaly ($\rho_0 - \rho_{min}$) and
506 U_m is the maximum streamwise velocity within the plume layer. The entrainment rate E
507 from the present experiments is plotted against bulk Richardson number in Figure 9a, along
508 with those of Ellison and Turner (1959); Chu and Vanvari (1976); Pedersen (1980); Buch
509 (1980), as summarized by Christodoulou (1986). Our data are in the low Ri_b regime and are
510 in good agreement with data from Ellison and Turner (1959) in a similar Ri_b range. Two
511 low Fr_i runs have significantly lower entrainment rates (diamonds in Figure 9a inset). These
512 correspond to very thin buoyant layers and we hypothesize that they may be in a different
513 regime than the other runs as described previously. Excluding those two points, data from
514 the present experiments follow a $E = k_1 Ri_b^{-1/2}$ relationship in the region of $Ri_b < O(1)$,
515 where in the present experiment $k_1 = 0.02$. This Ri_b region is described by Christodoulou
516 (1986) as the region where the mixing takes place through vortex entrainment.

517 It is also valuable to investigate the dependence of E on Fr_i , which is an independent
518 parameter describing the strength of the inflow as opposed to Ri_b which characterizes the
519 sheared flow observed in situ. We observe a clear linear relationship between E and Fr_i for
520 all experiments (Figure 9b). The data shows no significant difference in entrainment rate
521 between spreading and channelized cases, suggesting again that local mixing is not modified
522 by lateral spreading. Entrainment parameterizations based on the bulk Richardson number
523 such as Ellison and Turner (1959) require a priori knowledge of the current properties,
524 which makes the analytical theory complex or even unsolvable (Hetland 2010). Simpler
525 parameterizations, such as $E = \text{constant}$ or $w_e = \text{constant}$ have also been commonly used in

526 numerical modeling, but do not capture the dependence of mixing on inflow conditions. Our
 527 result indicates a simple relation between the entrainment rate and inflow Froude number,
 528 $E = aFr_i$. This suggests Fr_i controls the total energy input to the plume system, and
 529 highlights the usefulness of using Fr_i to predict the amount of entrainment and mixing in
 530 the near field plume region.

531 Moreover, the relationship between Froude number and the bulk Richardson number
 532 ($Ri_b = 1/Fr^2$) shows that the inflow bulk Richardson number (here we call it Ri_{bi}) also
 533 fits the relationship suggested by Christodoulou (1986): $E = k_2 Ri_{bi}^{-1/2}$, where $k_2 = 0.03$.
 534 Although both in-situ and inflow Richardson numbers agree with the empirical relationship
 535 between entrainment rate and Richardson number, the in-situ measurement shows more
 536 scatter and weaker fit. Note that there are errors in measuring the density and velocity
 537 field and calculating the plume thickness (H_p) and hence Ri_b in the plume layer, which may
 538 explain some of the difference. More importantly, however, the in-situ calculation of Ri_b uses
 539 the velocity and density at the end of the field of view, while most of the mixing happens
 540 near the lift-off region. The inflow Richardson number (or Froude number) describes the
 541 initial conditions at the inflow and may represent the potential for mixing. This appears to
 542 be a better way to predict the actual amount of entrainment.

543 *f. Turbulent buoyancy flux*

544 The turbulent buoyancy flux is a direct measure of mixing due to turbulence that can be
 545 estimated from the available data using the control volume method (MacDonald and Geyer
 546 2004). It is defined as:

$$B = \frac{g}{\rho_0} \overline{\rho'w'} = \frac{g}{\rho_0} (\overline{\rho w} - \bar{\rho} \bar{w}). \quad (11)$$

547 where $\frac{g}{\rho_0} \overline{\rho w}$ and $\frac{g}{\rho_0} \bar{\rho} \bar{w}$ are the total and mean vertical density fluxes through isohalines,
 548 respectively. The mean flux is the same order of magnitude as the total and the turbulent

549 buoyancy flux is approximately two orders of magnitude smaller. In all cases, the shapes of
 550 the turbulent buoyancy flux profiles are relatively similar (Figure 10). The peak buoyancy
 551 flux occurs approximately one quarter of the plume thickness below the surface in spreading
 552 cases and slightly lower than half the plume thickness in channelized cases. All profiles
 553 decrease to zero at the ambient water interface, and are forced to be zero at the water
 554 surface. This profile shape is consistent with field observations from the Fraser (MacDonald
 555 and Geyer 2004) and Merrimack (MacDonald et al. 2007) river plumes. The buoyancy flux
 556 profiles generally have a higher peak in the channelized cases than in the spreading cases,
 557 especially for the high flow rate runs. In the channelized cases the turbulent buoyancy flux
 558 decreases to zero right at the plume base ($z/H_p = -1$), while in the spreading cases it
 559 decreases to zero around $z/H_p = -0.8$.

560 The depth averaged buoyancy flux over the plume layer is defined as $\bar{B} = \frac{1}{H_p} \int_{-H_p}^0 B(z) dz$.
 561 The average flux is then normalized to form the mixing parameter $\xi = \frac{\bar{B}}{\Delta u g}$, where Δu is the
 562 velocity difference between upper and lower layers. In our case, $\Delta u = U_m$ because there is no
 563 velocity in the ambient water. Because the shapes of the profiles are similar, the relationship
 564 between B values in different experiments is similar if we use the maximum flux B_{\max} instead
 565 of the depth averaged value \bar{B} . The value of ξ increases with Fr_i as expected (Figure 11a).
 566 Most importantly, however, there is no difference between the values of ξ in spreading and
 567 channelized cases (Figure 11a). The average value of ξ is $(1.0 \pm 0.7) \times 10^{-4}$ in the spreading
 568 cases and $(1.4 \pm 1.0) \times 10^{-4}$ in the channelized cases. This somewhat surprising result is
 569 evidence that local turbulent mixing processes are unaffected by plume spreading.

570 One significant difference between spreading and channelized cases is that the horizontal
 571 surface area of the plume is much higher in spreading cases (Figure 4a). The plume area is
 572 two to six times larger in the spreading cases than in the channelized cases (Figure 11b). In
 573 order to account for effect of interfacial area on the total mixing, we define the area-integrated
 574 turbulent buoyancy flux $\xi_A = \xi \frac{A_S}{A_C}$, where A_S and A_C are the horizontal surface areas in a
 575 given spreading run and the corresponding channelized run with similar inflow conditions,

576 respectively. To compare, the area-integrated turbulent buoyancy flux in channelized runs
577 is defined as $\xi_A = \xi_{AC}^{AC}$. In the spreading runs ξ_A is about $2 \sim 6$ times larger than the
578 channelized cases (Figure 11c). The average value of ξ_A is $(4.4 \pm 3.3) \times 10^{-4}$ in the spreading
579 cases and $(1.4 \pm 1.2) \times 10^{-4}$ in the channelized cases. Thus spreading increases the total
580 turbulent mixing, even though it does not appear to change the local mixing processes.

581 4. Discussion

582 The results from the present experiments support the conclusion that lateral spread-
583 ing significantly modifies the plume's vertical structure (Figure 6) and the plume density
584 anomaly (Figure 7). However, these experiments do not support the hypothesis that lateral
585 spreading increases the local mixing, as quantified by the entrainment rate (Figure 9b) or
586 the turbulent buoyancy flux (Figure 11a). We observed increased mixing in the spreading
587 cases compared with the channelized experiments (Figure 11c), but this is shown to be the
588 result of the increased interfacial surface area of the spreading plumes (Figure 11b), rather
589 than any impact of spreading on the local mixing processes.

590 The results of the present experiments motivate the following question: why doesn't
591 lateral spreading impact turbulent mixing in the plume, even when it significantly modifies
592 the vertical plume structure? We investigate two possible mechanisms that may explain this
593 result in §4.b. Prior to this, we compare the non-dimensional mixing scales observed in the
594 experiments with those of full-scale plume in order to investigate the applicability of the
595 results.

596 *a. Applicability of the laboratory experiments to river plumes*

597 Here we use three non-dimensional parameters for comparing the turbulent mixing in the
598 laboratory simulations, field observations and numerical models:

$$I = \frac{\varepsilon}{\nu N^2}; \quad \eta = \frac{L_o}{H_p}; \quad \xi = \frac{\bar{B}}{\Delta u g'}; \quad (12)$$

599 The first parameter, I , commonly referred to as the buoyancy Reynolds number, can
600 be interpreted as the ratio of the destabilizing effects of turbulent stirring to the stabilizing
601 effects resulting from the combined action of buoyancy and viscosity (Ivey et al. 2008). When
602 I is above a threshold value of 20–30 the turbulence in the stratified fluid can be maintained
603 (Stillinger et al. 1983). Shih et al. (2005) suggest three discernible regimes of turbulent flow
604 based on their DNS results: a diffusive regime where $I < 7$; an intermediate regime where
605 $7 < I < 100$; and an energetic regime where $I > 100$.

606 The turbulent kinetic energy dissipation rate ε could not be measured directly in our
607 experiments. In order to estimate an approximate value for ε we assume the turbulent field
608 is homogeneous and isotropic and the flux Richardson number $Ri_f = B/P$ is at its maximum
609 value 0.2. This assumption is valid in stratified shear flow, when the Kelvin-Helmholtz billows
610 are the primary mechanism of turbulence generation (MacDonald and Geyer 2004). With
611 these assumptions we can estimate ε based on the difference between the production and
612 buoyancy flux, ($\varepsilon = \frac{1}{1-Ri_f}\bar{B}$). The values in our experiments range from 10^{-5} to 10^{-4} m²/s³,
613 which are typical of observations in stratified coastal environments (Orton and Jay 2005;
614 MacDonald et al. 2007) and one order of magnitude smaller than the lift-off zone in Fraser
615 River plume (MacDonald and Geyer 2004).

616 The present results for I are plotted in figure 12a, with two thresholds delineating the
617 three regimes indicated by a dashed line ($I = 10$) and dash-dot line ($I = 100$). One of the
618 low Fr_i spreading runs falls in the diffusive regime where the flow is mostly dominated by
619 molecular diffusivity. Most of the data are in the transition or energetic regime in the range
620 of $10 < I < 1000$. They are slightly higher than previous laboratory experiments by Ivey and
621 Imberger (1991) of I between approximately 10 and 100. Field observations have suggested
622 values of I are on the order of 10^4 to 10^5 in the highly stratified Columbia River estuary
623 during ebb tide (Kay and Jay 2003) and in the Fraser River plume (MacDonald and Geyer

2004). In the Connecticut River estuary Geyer et al. (2010) estimated that $I = 100 - 500$. Thus the range of values of I achieved in the laboratory experiments are on the low end of the range observed in natural systems. More importantly, the values that we obtain are in the transitional and energetic regimes where we would expect the processes to be similar to larger-scale systems.

Previous studies have suggested that mixing in gravity driven currents is accomplished through the generation of Kelvin-Helmholtz instabilities as density overturns with scales of similar size to the Ozmidov scale, $L_o = (\varepsilon/N^3)^{1/2}$ (MacDonald and Geyer 2004). These overturns are clearly seen in the mixing layer of the plume in both cases (Figure 2 and 3). An important question for the current experiments is whether the thinness of the plume layer limits the scales of turbulent mixing and, in particular, if this effect could differentially influence the mixing in the thinner spreading plume runs. Figure 12b shows estimates of L_o normalized by the plume thickness H_p . The Ozmidov scale L_o is roughly the same size as the plume thickness H_p , i.e., L_o/H_p is on the order of 1. We observe that L_o/H_p increases with Fr_i and that L_o exceeds H_p in a few high Fr_i experiments. This indicates that the plume thickness has the potential to inhibit the turbulence. However, there is no indication that the influence of the plume thickness is greater in the spreading or channelized experiments. Thus we conclude that depth limitation cannot explain the result that the local mixing processes are unaffected by spreading.

Finally, the mixing parameter ξ represents the efficiency with which energy is extracted from the mean flow and converted into turbulent energy (MacDonald and Geyer 2004; MacDonald and Chen 2012). The values of ξ observed in the present experiments are in the range from 10^{-5} to 10^{-3} (Figure 11a). This broad range agrees with results from the Fraser River, where ξ was estimated to be approximately 2.6×10^{-4} (MacDonald and Geyer 2004). The channelized cases can be compared with the non-spreading limit $\xi_0 = 5 \times 10^{-5}$ described in MacDonald and Chen (2012).

Although the Reynolds numbers of the laboratory-generated plumes (Table 1) are sig-

651 nificantly lower than most plumes observed in the field, the buoyancy Reynolds number I
652 and the mixing parameter ξ are both in the same approximate range, suggesting that the
653 turbulence is sufficiently active to represent the processes observed in the field. So we con-
654 clude that Reynolds number limitation does not appear to influence our findings from the
655 laboratory experiments.

656 *b. Two possible mechanisms*

657 The results presented in §3.c describe important differences in the structure of channelized
658 and spreading currents. The buoyant current in the channelized case has step-wise density
659 and velocity structures, as is commonly observed in the two-layer exchange flow in estuary
660 channels (Figure 13a line in the dark gray area). It has a minimum density (maximum
661 velocity) in the near-surface uniform density layer, a maximum density (zero velocity) in
662 the bottom layer and a mixing layer in between in which the density increases and velocity
663 decreases continuously. In the spreading case, however, the near-surface uniform density
664 layer disappears and the vertical plume structure consists of a 1–1/2 layer system with a
665 mixing layer that extends approximately linearly to the water surface (Figure 13a line in the
666 light gray area). This structure is consistent with field observations in near-field river plumes
667 (e.g. MacDonald and Geyer (2004); MacDonald et al. (2007); Kilcher et al. (2012)). Despite
668 the difference in the structure of the spreading and channelized currents, the properties of
669 the mixing layer such as the mixing layer thickness, the density structure and the velocity
670 structure are almost identical in both cases for similar inflow conditions. In addition, mixing
671 properties such as the entrainment velocity and turbulent buoyancy flux are similar in the
672 spreading and channelized cases.

673 We consider two mechanisms that may explain the unexpected result that local mixing
674 processes are not affected by plume spreading. Both invoke the idea that there is vertical
675 structure to the spreading and mixing processes and that the influence of spreading on mixing
676 will be minimized if the two processes are misaligned spatially. The proposed mechanisms

677 leading to this misalignment are different in the region very near the river mouth and in
678 the rest of the near-field plume. They are described below and summarized schematically
679 in Figure 13. First, in the jet-to-plume transition region immediately offshore of the river
680 mouth where the current transforms from a buoyant jet into a river plume, the spreading
681 occurs primarily in the near-surface uniform density region while most mixing occurs at the
682 plume base. Second, although the spreading and mixing occur within the same layer in the
683 near field region, their structures are offset such that spreading does not significantly impact
684 the region of maximum mixing.

685 1) JET-TO-PLUME REGION

686 The region near the river mouth is one of the most energetic regions in the river plume.
687 Total mixing in this region is of the same order of magnitude as it is in the mid-field and far-
688 field plumes, even though the near-field is orders of magnitude smaller in area (Hetland and
689 MacDonald 2008). In the spreading case, the flow evolves in this region from a channelized
690 current in the estuary into a buoyant river plume. As the current moves offshore, the density
691 and velocity profiles change dramatically from two-layer step-wise profiles to mixing layer
692 profiles (Figure 13a).

693 Lateral spreading is due to the horizontal pressure gradient, which is highest at the water
694 surface and decreases to zero at the plume base. Our hypothesis is that in this transition
695 region the spreading occurs preferentially in the near-surface uniform density layer (Figure
696 13c dark gray area) while the most energetic mixing occurs at the lower layer of the plume
697 (Figure 13c light gray area). Thus, the mixing layer does not experience significant lateral
698 spreading. Instead, the main result of lateral spreading is that the near-surface uniform
699 density layer disappears, as shown in the cartoon of this jet-plume region (Figure 13c).

700 Jirka et al. (1981) describe the jet-plume region as the region where momentum dominates
701 over buoyancy and the impact of the initial channel geometry disappears. They define a jet-
702 plume length scale as $L_M = (Q_0 U_0)^{3/4} / (Q_0 g'_0)^{1/2}$. The jet-plume length scale is relatively

703 small compared to the whole plume and is even smaller than the near-field plume scale. In
 704 this region the plume behaves much like a buoyant jet, driven by the enhanced velocities of
 705 the discharge as it initially enters the coastal region.

706 In our experiments, the jet-to-plume length scale is 8.8 cm for $g' = 2 \text{ cm/s}^2$ (3.5 cm for
 707 $g' = 5 \text{ cm/s}^2$). Thus, the location of our PIV-PLIF measurements, which is ≈ 30 cm from
 708 the river mouth, is well beyond the jet-to-plume region and we would expect to observe
 709 only the fully developed mixing layer profiles. This is consistent with the observed profiles
 710 shown in Figure 6. We suggest that spreading within the jet-to-plume region is due primarily
 711 to lateral slumping of the initially uniform density surface layer and that mixing, which is
 712 primarily at the base of this layer is unaffected.

713 2) NEAR-FIELD REGION

714 Seaward of the jet-to-plume region, the spreading plumes are characterized by linear
 715 density and velocity profiles (Figure 6a). Profiles of vertical buoyancy flux in the spreading
 716 cases have maxima at $z \approx -H_p/4$ (Figure 10) and decrease to zero both at the surface and
 717 at the plume base.

718 As discussed in the §4.b.1, lateral spreading is due to a horizontal pressure gradient
 719 normal to the plume axis. The pressure gradient (dP/dy) is expected to be a linear function
 720 of depth within the plume; highest at the water surface and decreasing to zero at the plume
 721 base. Although we cannot directly calculate the spreading rate based on the horizontal
 722 pressure gradient, it is reasonable to assume the spreading rate should have the same shape
 723 as the pressure gradient.

724 In addition, we can calculate the local lateral spreading rate (dv/dy) along the plume
 725 axis directly from the measured velocity field based on the continuity equation:

$$\frac{dv}{dy} = -\left(\frac{du}{dx} + \frac{dw}{dz}\right). \quad (13)$$

726 The local lateral spreading profile is calculated by averaging the dv/dy field horizontally
727 over 12 cm and then normalizing it by its maximum value. In Figure 14 the normalized
728 spreading rate is plotted against the normalized vertical axis and fitted by an exponential
729 curve to compare with the normalized buoyancy flux profiles. The lateral spreading rate
730 has its maximum value at the water surface and is highest within the top half of the layer.
731 It decreases dramatically in the lower half of plume layer. Meanwhile, the mixing profile
732 has its maximum at $z \approx -H_p/4$ and decreases both upward and downward. The spreading
733 rate at the location of the maximum buoyancy flux is between 20% to 50% of the maximum
734 spreading rate at the surface. Throughout the lower three-quarters of plume layer lateral
735 spreading is significantly reduced and is not expected to influence mixing. This lower three-
736 quarters of the layer is where the entrainment generates the greatest turbulent buoyancy
737 flux because the density gradient is high, i.e., entraining dense fluid from the ambient water
738 ($w_e > 0$) in Figure 8.

739 This mechanism is similar to the mechanism described for the jet-to-plume region; the
740 maxima in the spreading and mixing profiles are not coincident. The spreading at the top
741 unmixed layer and mixing (entrainment) at the plume base with no overlap in the jet-to-
742 plume region is the extreme case of the offset mixing and spreading profiles discussed here
743 (Figure 13b).

744 The lateral spreading rate is commonly known to be related to the local internal gravity
745 wave speed, i.e., $\sqrt{g'h}$ (Wright and Coleman 1971; Hetland and MacDonald 2008). In our
746 experiments and in previous studies that use the control volume method, the lateral spreading
747 rate is assumed to be independent of depth (MacDonald and Geyer 2004; Kilcher et al. 2012).
748 This assumption agrees reasonably well with field data, although authors have hypothesized
749 that observed discrepancies may be attributed to the depth dependence in the spreading. Our
750 analysis supports this hypothesis; suggesting that the vertical structure of lateral spreading
751 rate actually may play an important role in determining the relationship between lateral
752 spreading and its effect on mixing.

5. Conclusion

This paper presents a direct comparison between channelized and freely spreading buoyant gravity currents with a continuous freshwater source. The configuration of the laboratory experiments simulates a coastal river inflow with a simplified geometry in order to better understand the role of lateral spreading on the mixing and dilution of river water as it enters the coastal ocean.

Consistent with predictions from previous work (Wright and Coleman 1971; Hetland and MacDonald 2008), we observe that the lateral spreading rate is highly dependent on the inflow condition as characterized by Fr_i : the plume is convergent when $Fr_i < 1$ and divergent when $Fr_i > 1$ (Figure 5). As a consequence of these changes to the spreading rate, the increase in plume area due to spreading within a given distance from the river mouth is significantly greater for low Fr_i than high Fr_i plumes (Figure 11b).

Lateral spreading dramatically modifies the plume's vertical structure; the spreading plumes consist of approximately linear density and velocity profiles that extend to the surface, whereas the channelized plumes have regions of uniform density and velocity near the surface (Figure 6). In addition, the average density of the plume layer at a fixed distance from the river mouth is higher in the spreading experiments than in the channelized experiments (Figure 7).

We estimate the entrainment rate E and the turbulent buoyancy flux B using the control volume method described by MacDonald and Geyer (2004). The entrainment rate is at the same order of magnitude as the previous laboratory studies. It fits the $Ri_b^{-1/2}$ law (Ellison and Turner 1959; Christodoulou 1986) in the low Ri_b region. A key outcome of this work is the observation that there is no difference in the entrainment rate or buoyancy flux between the channelized and spreading cases. This indicates that lateral spreading does not modify the local mixing efficiency, counter to the expectations outlined in MacDonald and Chen (2012). We hypothesize that this is because the spreading occurs preferentially near the surface, whereas buoyancy flux is greatest in the core of the current.

780 We conclude that spreading significantly increases the total mixing in the plume (Figure
781 11c). However, the increase in mixing is due to the increase in the area of the plume (Figure
782 11b) as opposed to changes in the local mixing processes associated with spreading (Figure
783 11a). As estuary water enters the coastal ocean, lateral expansion occurs preferentially near
784 the surface, eliminating the uniform density layer observed at the surface in the estuary
785 and shifting the mixing layer upwards to the water surface. Near-surface water is thus
786 redistributed across a much wider area, where it then forms the plume base and is susceptible
787 to mixing. The result of the lateral advection of fresh near-surface water is that the plume
788 layer is more diluted on average in the presence of spreading than an equivalent channelized
789 flow.

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TABLE 1. The parameters of the experiments.

	$g'_0(\text{cm}^2/\text{s})$	$Q_0(\text{gpm})$	$U_0(\text{cm}/\text{s})$	Fr_i	Re
SP1	2.11	6	2.52	0.78	7568
SP2	2.01	9	3.78	1.19	11352
SP3	1.80	12	5.05	1.68	15136
SP4	1.74	15	6.31	2.14	18920
SP5	5.05	6	2.52	0.50	7568
SP6	5.28	9	3.88	0.74	11352
SP7	5.40	12	5.05	0.97	15136
SP8	5.64	15	6.31	1.19	18920
CH1	2.13	6	2.52	0.73	7277
CH2	2.35	9	3.78	1.29	12613
CH3	2.22	12	5.05	1.43	14554
CH4	2.34	15	6.31	1.84	18920
CH5	5.38	6	2.52	0.49	7568
CH6	5.20	9	3.78	0.79	11825
CH7	4.97	12	5.05	1.01	15136
CH8	5.02	15	6.31	1.26	18920
PL1	2.40	5	2.15	0.62	6459
PL2	2.01	9	3.88	1.19	11352
PL3	2.64	12	5.05	1.39	15080
PL4	1.80	12	5.05	1.68	15136
PL5	9.13	4	1.68	0.25	5045
PL6	5.05	6	2.52	0.5	7568
PL7	4.05	6	2.52	0.56	7532
PL8	5.27	8	3.44	0.67	10334
PL9	5.28	9	3.88	0.74	11352
PL10	5.64	15	6.31	1.19	18920

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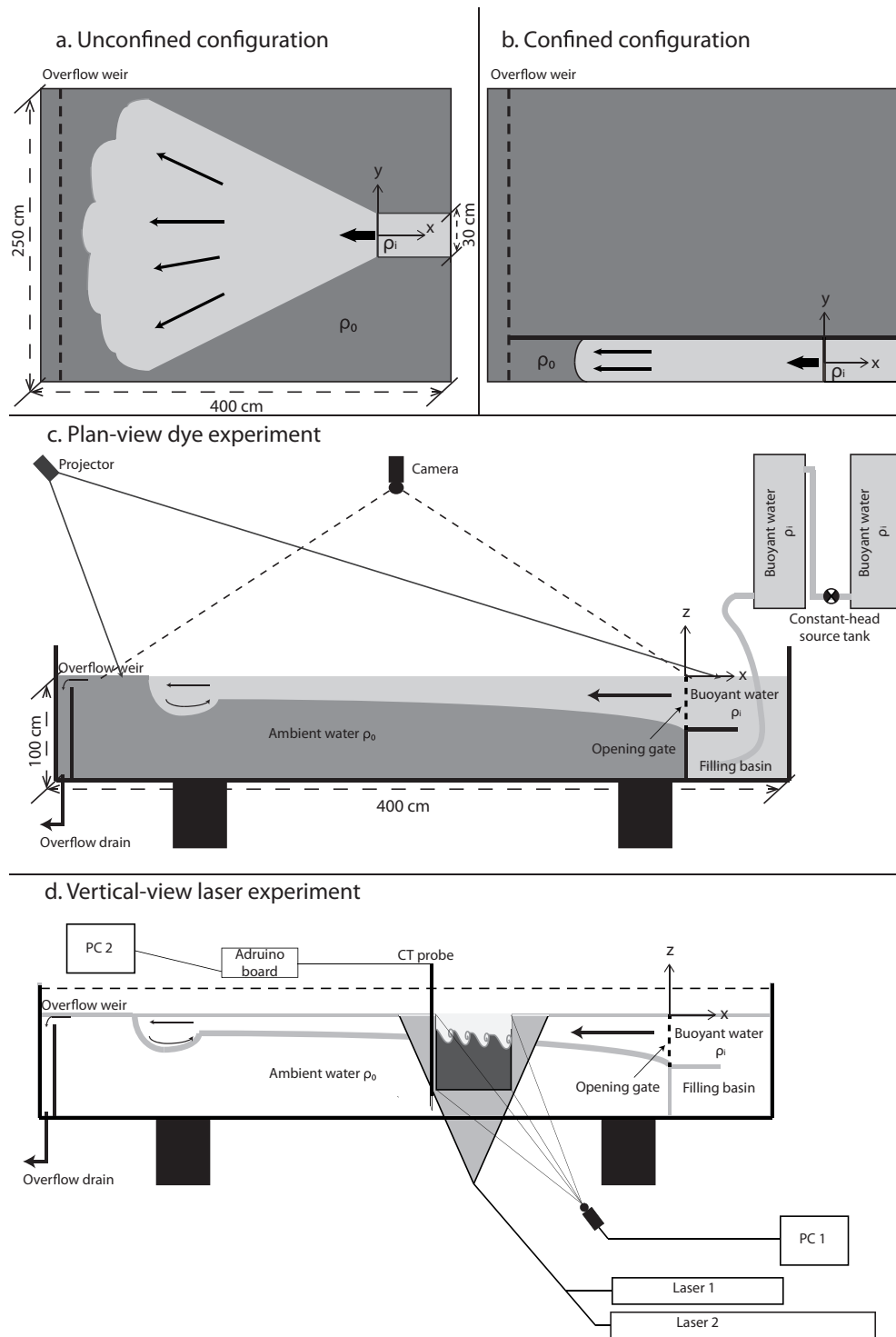


FIG. 1. Schematic of laterally a) unconfined (spreading) and b) confined (channelized) gravity currents and schematics of experimental facility and instrumentation for c) the plan-view dye experiments and d) vertical-view laser experiments. The Cartesian coordinate is defined in the schematic: origin is at the center of the river mouth water surface, x is the onshore coordinate, y is the alongshore coordinate, z is the vertical coordinate with positive upward.

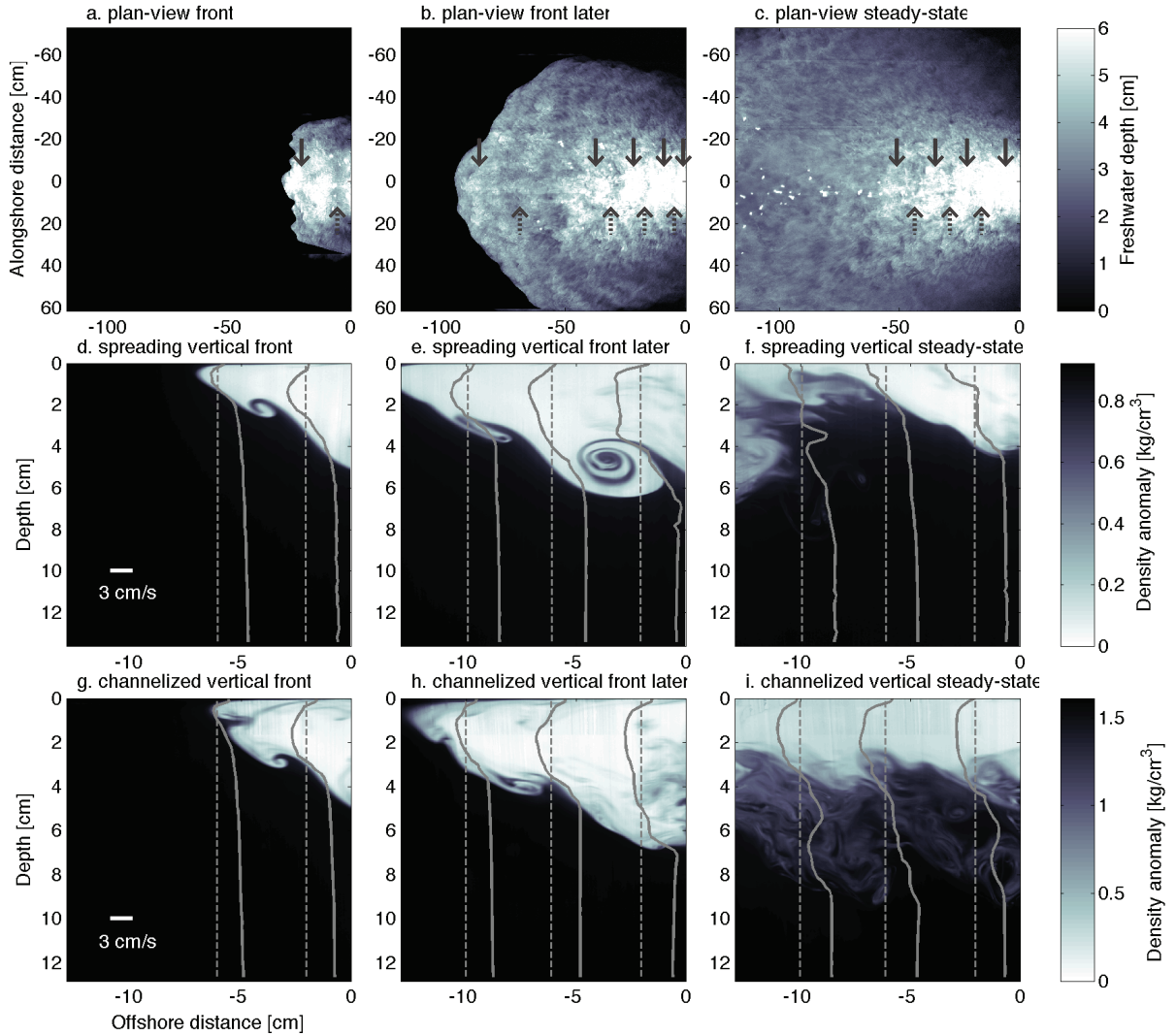


FIG. 2. High Fr_i spreading ($Fr_i = 1.68$) and channelized ($Fr_i = 1.43$) runs. Plan-view freshwater thickness fields from the spreading run PL4 (a-c), density field with superimposed front-relative velocity profiles from the spreading run SP3 (d-f), and density field with superimposed front-relative velocity profiles from the corresponding channelized run CH3 (g-i). Solid arrows in subplots a-c point the frontal bore and brighter bands, while dash arrows point darker bands. Dash lines in d-i are zero velocity lines and velocity scale bars are in subplots d and g.

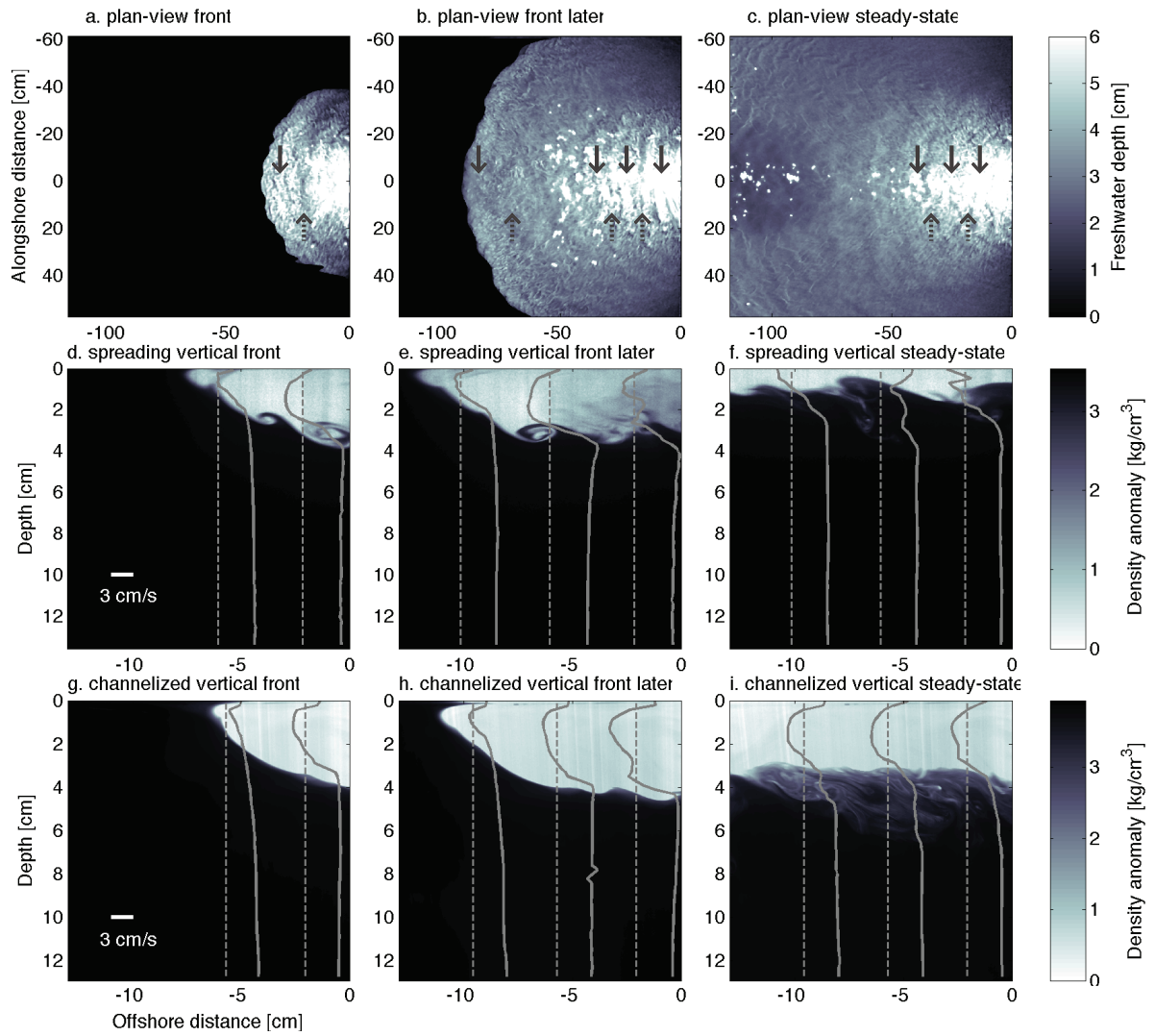


FIG. 3. Same as Figure 2 but for low Fr_i runs (SP5/PL6: $Fr_i = 0.50$ and CH5: $Fr_i = 0.49$).

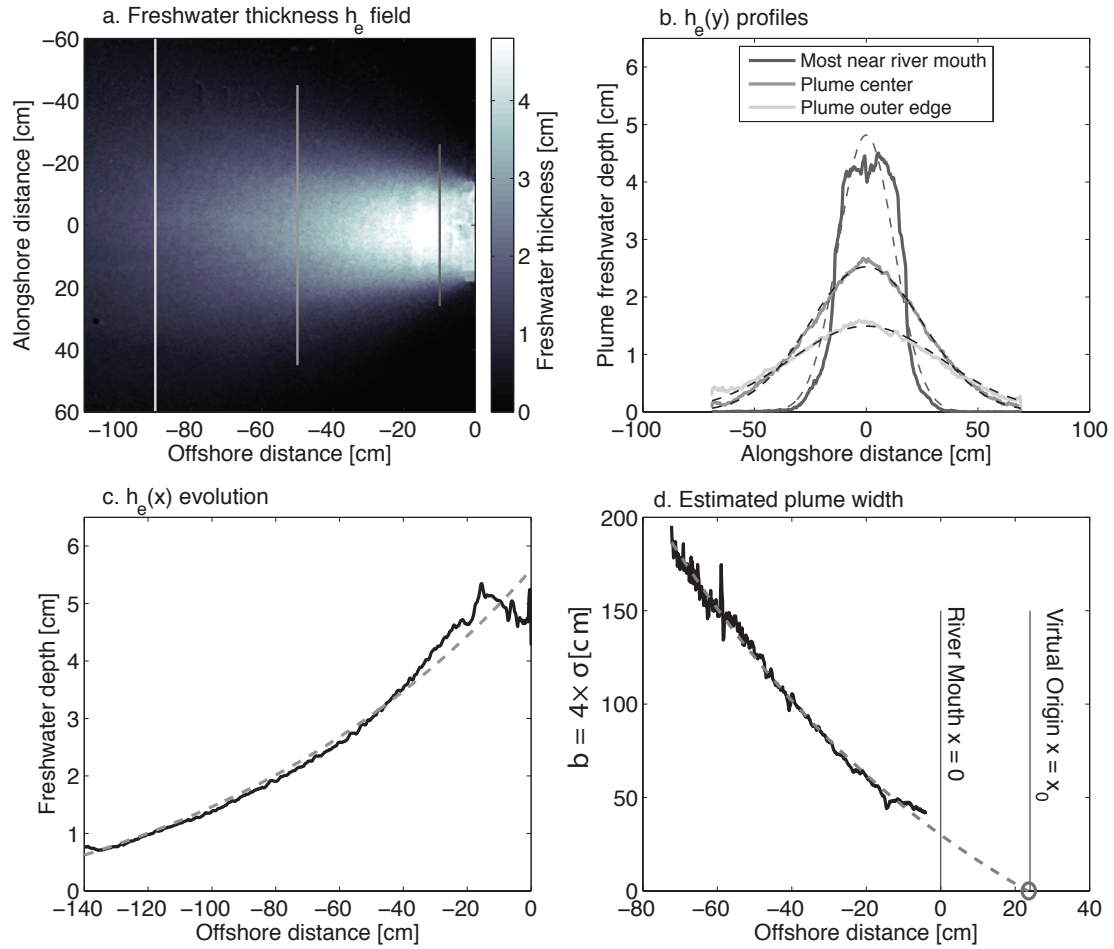


FIG. 4. a) Time averaged plan-view freshwater thickness h_e field, b) $h_e(y)$ profiles at the three cross-sections shown in panel a) (dashed lines are the Gaussian fit to each profiles), c) $h_e(x)$ evolution along the plume center line ($y = 0$) (dash line is the exponential fit), and d) estimated plume width defined as $b = 4 \times \sigma$ based on Gaussian fit of $h_e(y)$ (dash line is the exponential fit) for a high Fr_i spreading run (PL4: $Fr_i = 1.68$; Figure 2a-c). The two black lines in panel d) indicate the river mouth $x = 0$ and the virtual origin $x = x_0$.

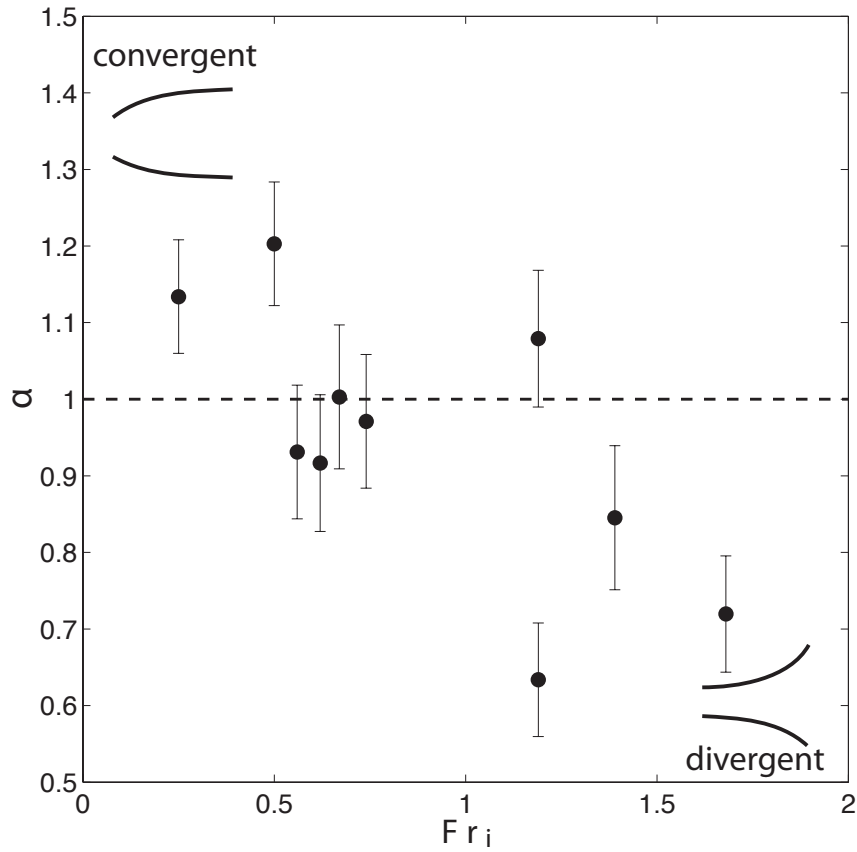


FIG. 5. Plot of spreading rate α vs. Fr_i . The dash line is $\alpha = 1$, indicating the pure radial spreading $u/\delta = x$. Schematic representations of convergent and divergent plumes, adopted from Hetland and MacDonald (2008), are shown above and below the $\alpha = 1$ line. Error bars are estimated based on the exponential fit to the width data.

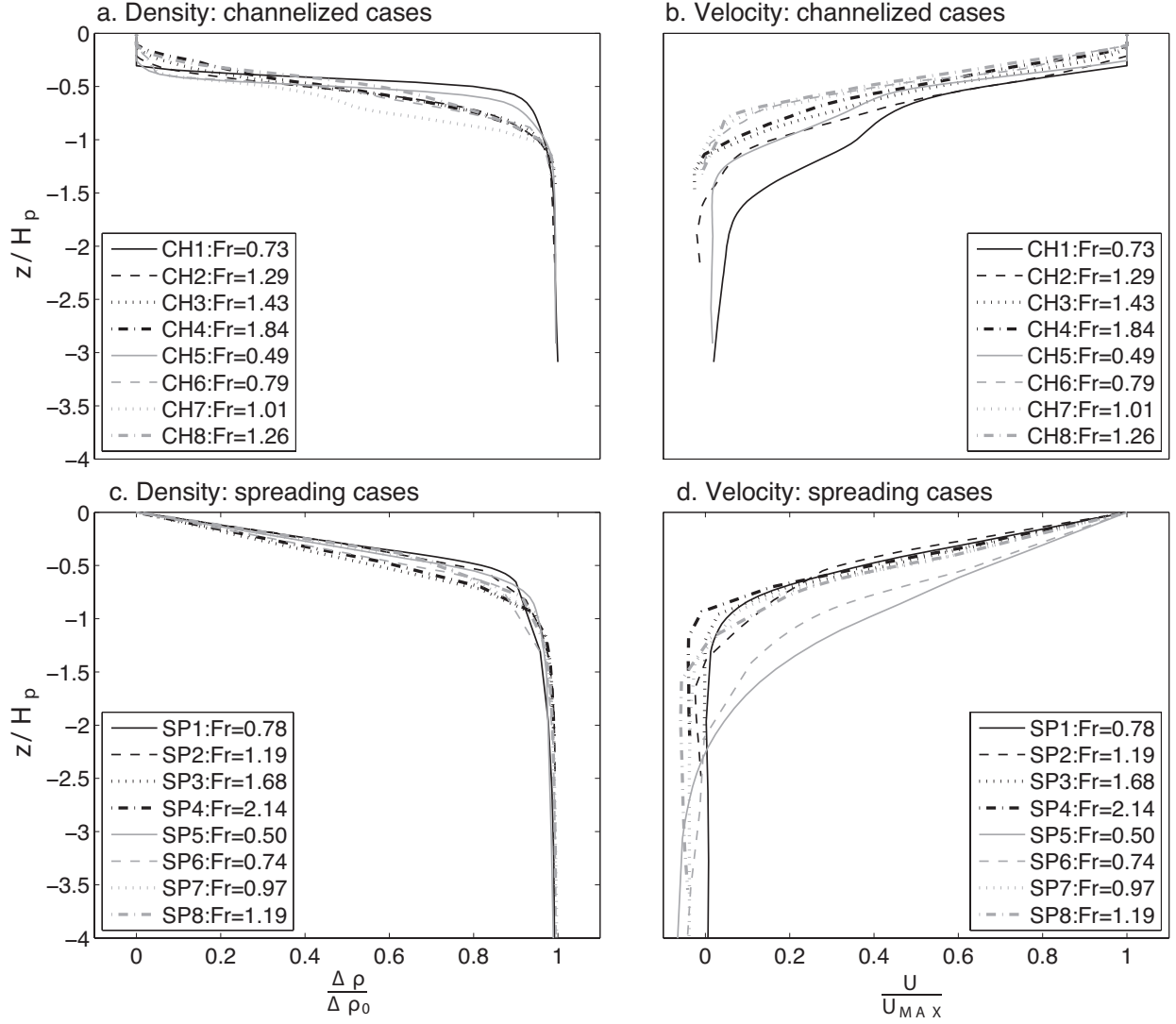


FIG. 6. Normalized density (a and c) and horizontal velocity (b and d) profiles for channelized (upper) and spreading (lower) runs. Detailed parameters of each runs shown in the legend are in Table 1. Vertical axis is normalized by the plume thickness (H_p), which is defined as the depth of the 95% freshwater flux contour.

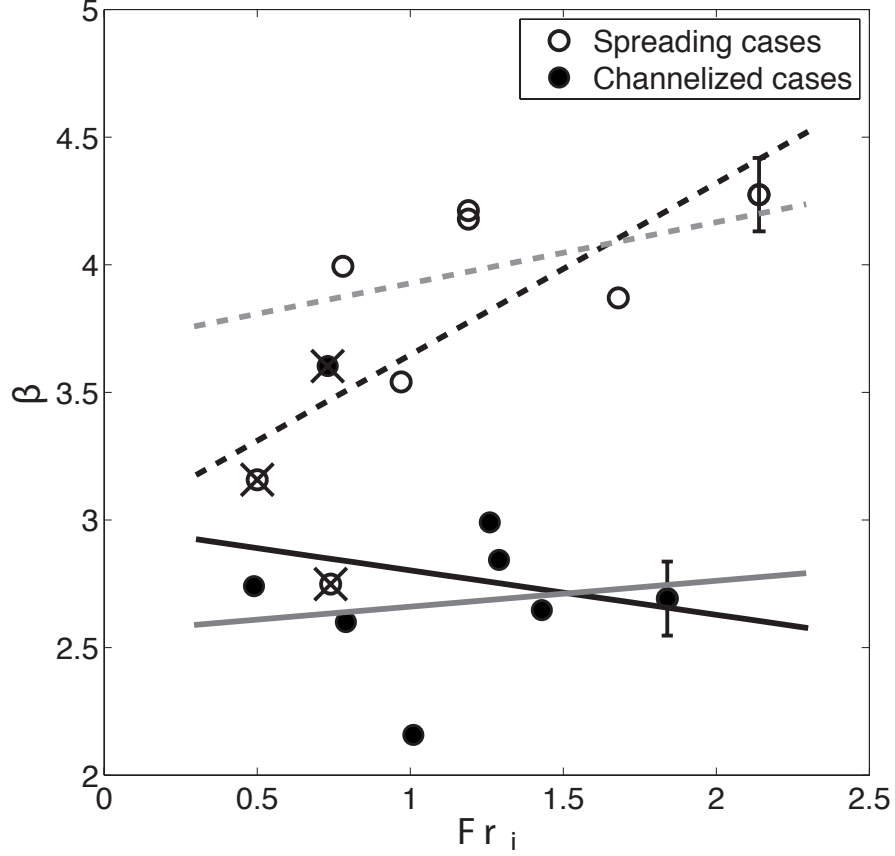


FIG. 7. Buoyancy anomaly (β) vs. Fr_i for spreading (open circles) and channelized (filled circles) runs. The averaged error bars for each configuration are plotted at the highest Fr_i points. Black dash line and black solid line are the linear fits to the spreading and channelized runs, respectively. Cross symbols highlight three points (SP5, SP6, and CH1) with opposite result comparing to other runs. Shaded dash line and shaded solid line are the linear fits without three anomalous points to the spreading and channelized runs, respectively. Two low Fr_i spreading runs (SP5 and SP6) have mis-matched density and velocity profiles (Figure 6). They are not within the same range as the other points in their group, and may reflect a different mixing regime. The reason for anomalous result of the low Fr_i channelized run (CH1) is unclear.

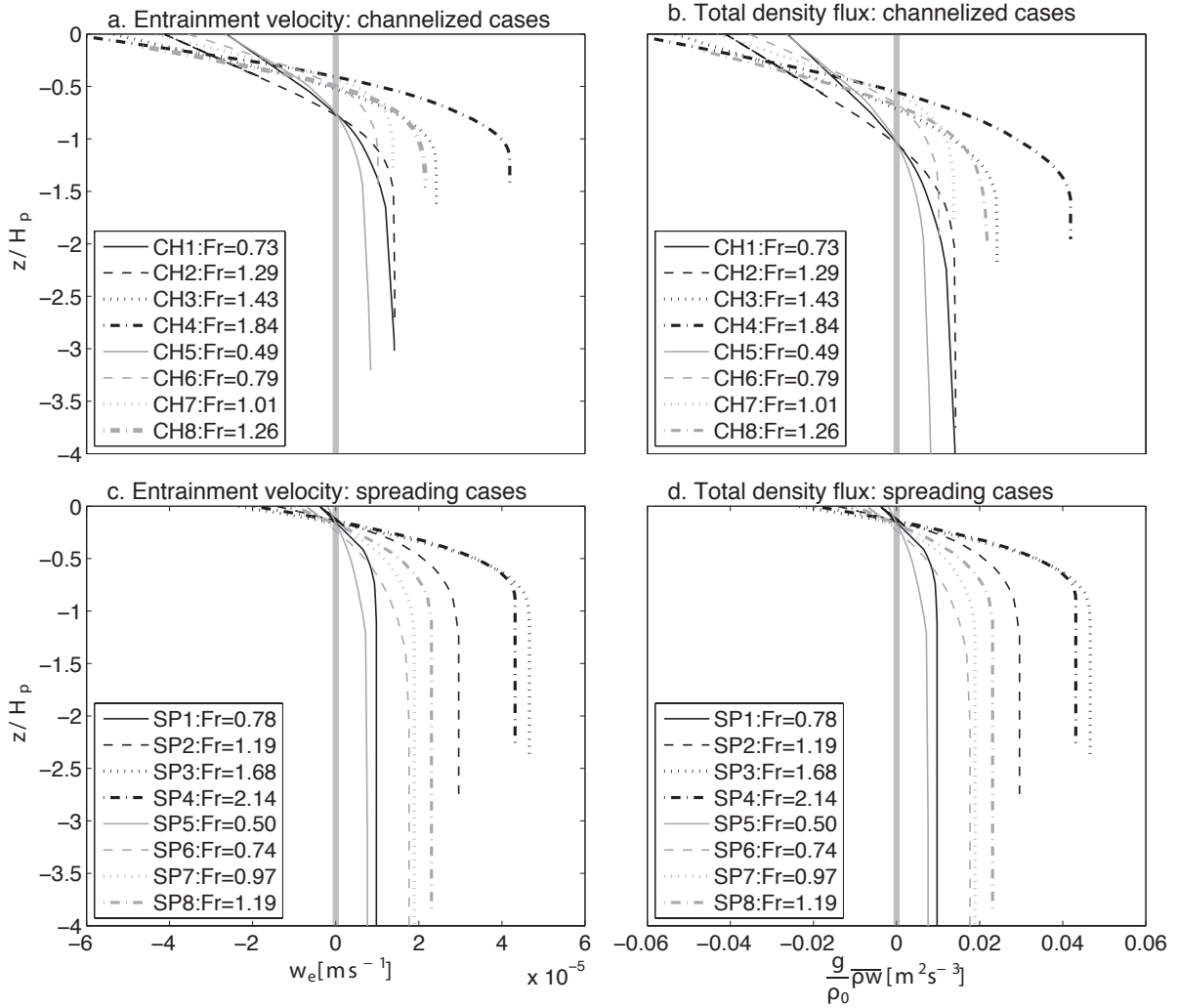


FIG. 8. Plume entrainment velocity w_e (a and c) and total vertical density flux $\frac{g}{\rho_0} \overline{\rho w}$ (b and d) for channelized (upper) and spreading (lower) experiments. The vertical thick shaded lines indicate the zero value of entrainment velocity or total vertical density flux in each panel.

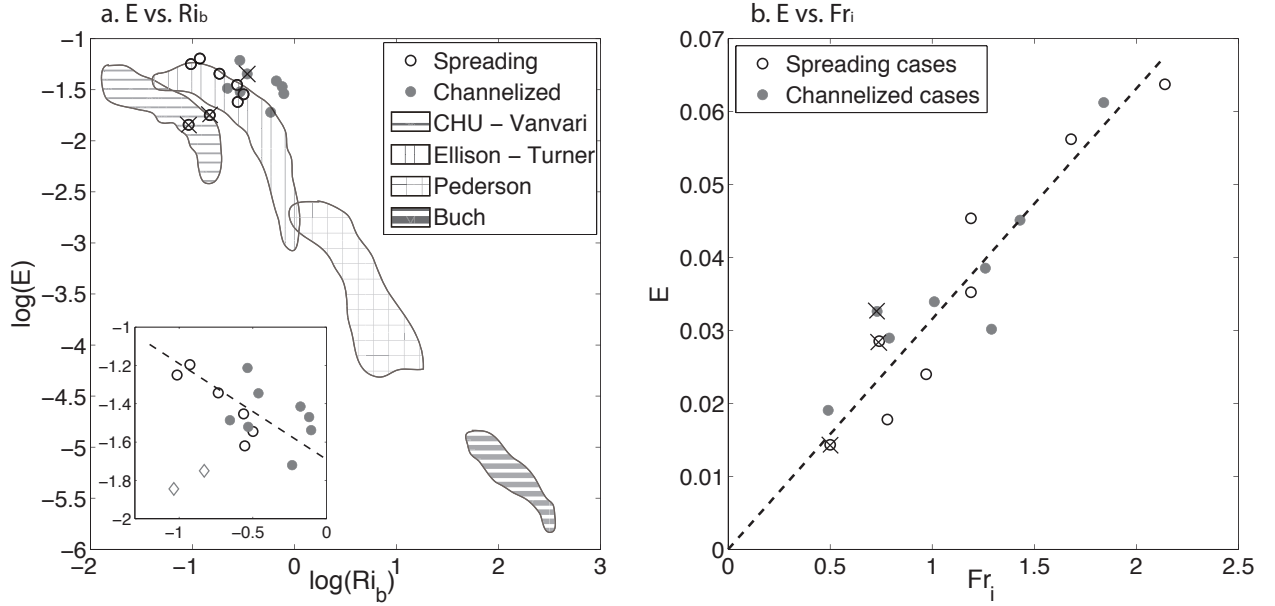


FIG. 9. a) Entrainment rate (E) vs. bulk Richardson number (Ri_b). Data represented by the shaded regions are drawn from Christodoulou (1986) with the data from laboratory experiments by Chu and Vanvari (1976); Ellison and Turner (1959); Pedersen (1980) and field observation by Buch (1980). The insert of (a) is the zoom-in of data from present experiments. The dash line is the fit to $E = 0.02 Ri_b^{-1/2}$ law to all data as suggested by Christodoulou (1986) (excluding the two low Fr_i runs indicated with diamonds). b) Entrainment rate (E) vs. inflow Froude number (Fr_i) for spreading and channelized cases. The dashed line corresponds to $E = 0.03 Ri_{bi}^{-1/2}$. Cross symbols highlight three points (SP5, SP6, and CH1) discussed in Figure 7.

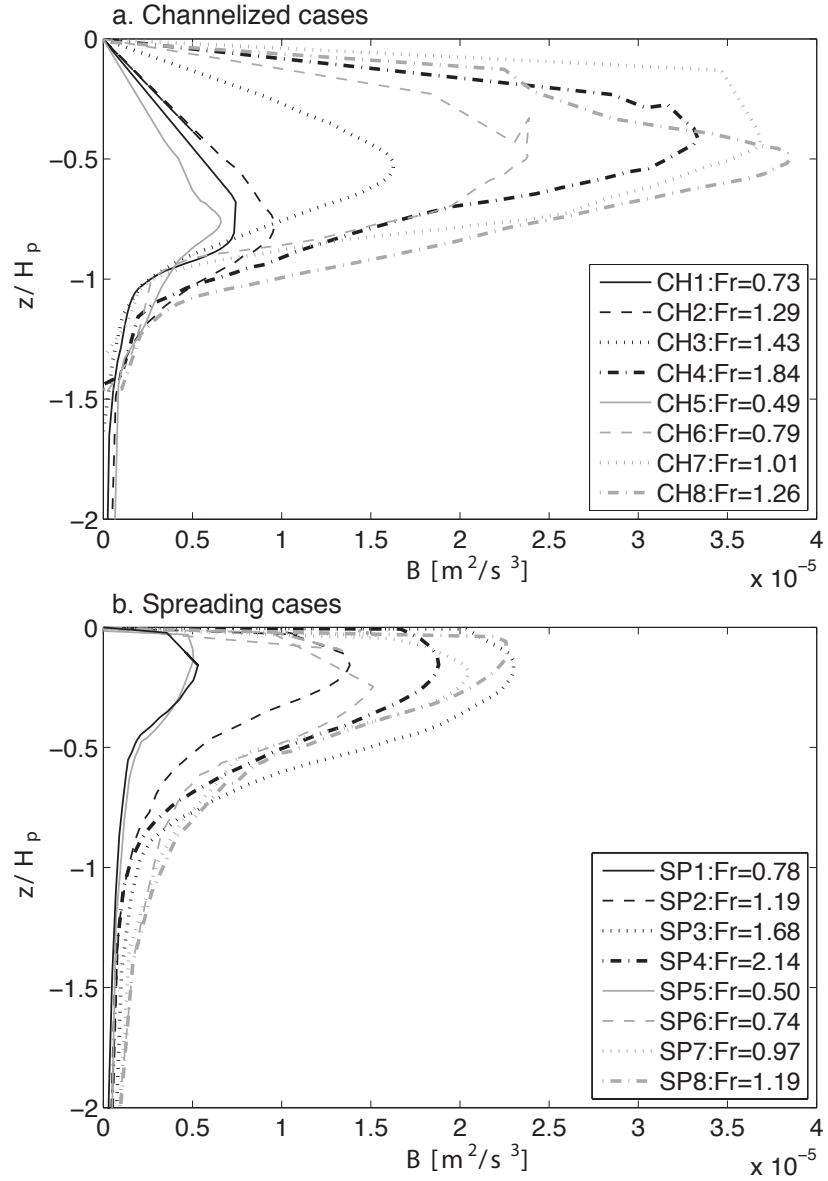


FIG. 10. Turbulent buoyancy flux profile estimated using the control volume method for a) channelized and b) spreading runs.

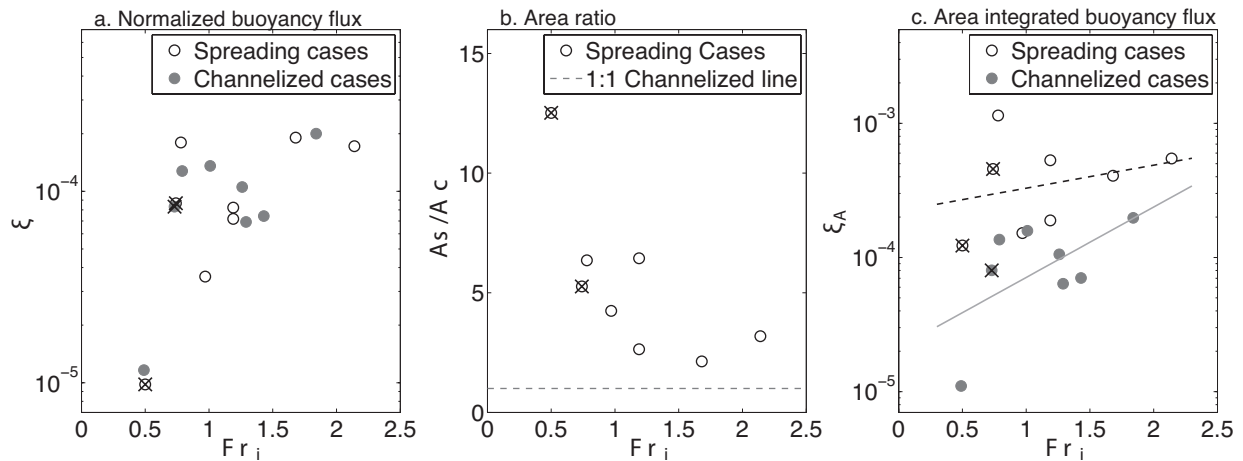


FIG. 11. a) Normalized buoyancy flux $\xi = \frac{\bar{B}}{\Delta u g'}$, b) ratio of plume area in spreading cases (A_S) to channelized cases (A_C), and c) area-integrated turbulent buoyancy flux ξ_A vs. Fr_i for spreading (open circles) and channelized (filled circles) runs. The area-integrated turbulent buoyancy flux is calculated as $\xi_A = \xi \frac{A_S}{A_C}$ in spreading runs and $\xi_A = \xi \frac{A_C}{A_C}$ in channelized runs. Linear fits in c) are applied separately for spreading (dash line) and for channelized runs (solid line). Cross symbols highlight three points (SP5, SP6, and CH1) that discussed in Figure 7.

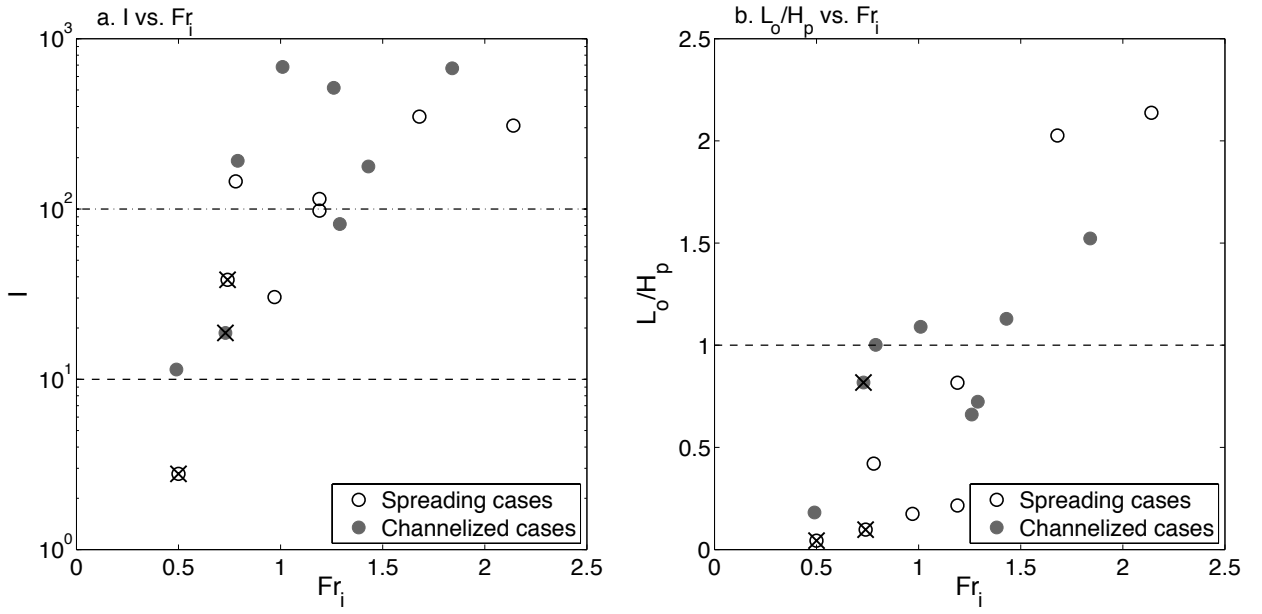


FIG. 12. a) $I = \varepsilon/\nu N^2$ vs. Fr_i , dash line and dash-dot lines are two thresholds for turbulent regime, $I = 10$ and $I = 100$. b) Ozmidov scale L_o normalized by plume thickness H_p vs. Fr_i . The dashed line is the reference for $L_o = H_p$. Cross symbols highlight three points (SP5, SP6, and CH1) that discussed in Figure 7.

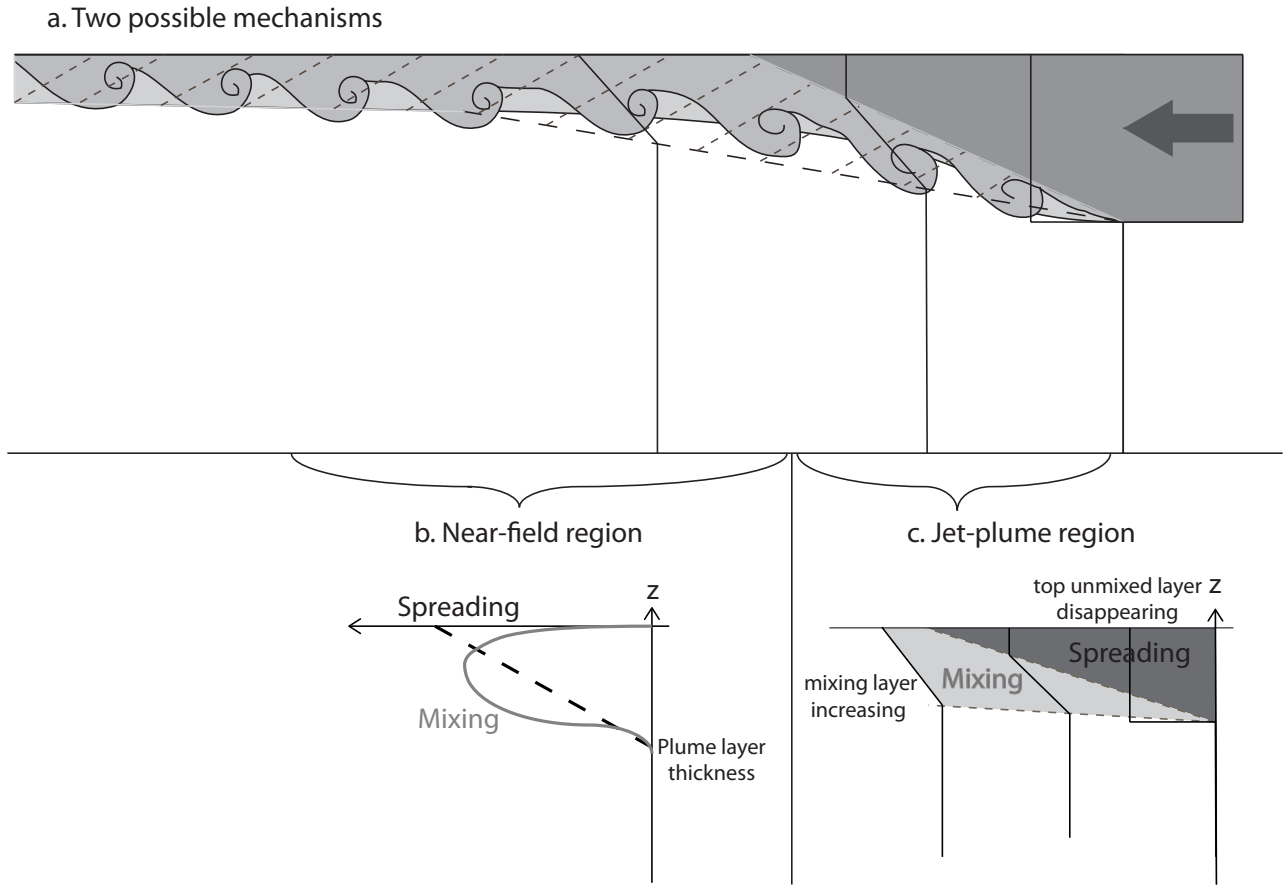


FIG. 13. Schematic representation of spreading and mixing in the jet-to-plume and near-field plume regions showing the transformation of vertical density structure (a). Three density (or velocity) profiles here represent the density profiles transforming from the fully channelized, transitional, and fully developed spreading regions, from right to left. Two possible mechanisms show the relationship between spreading and mixing in b) the near-field plume and c) the jet-plume region.

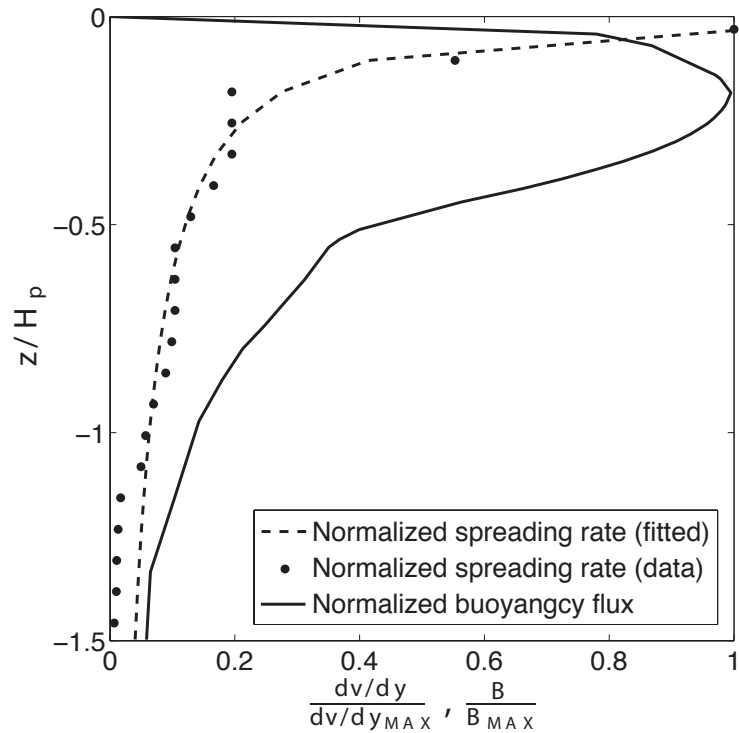


FIG. 14. Comparison of normalized spreading rate ($\frac{dv/dy}{dv/dy_{MAX}}$) profile with the normalized turbulent buoyancy flux ($\frac{B}{B_{MAX}}$) profile for an intermediate Froude number run (SP7:Fr=0.97). The dashed line is an exponential fit to the observed spreading rate profile.